

Section 5

Combined Heat & Power (Cogeneration)

Fundamentals of Turbines

Backpressure Turbines

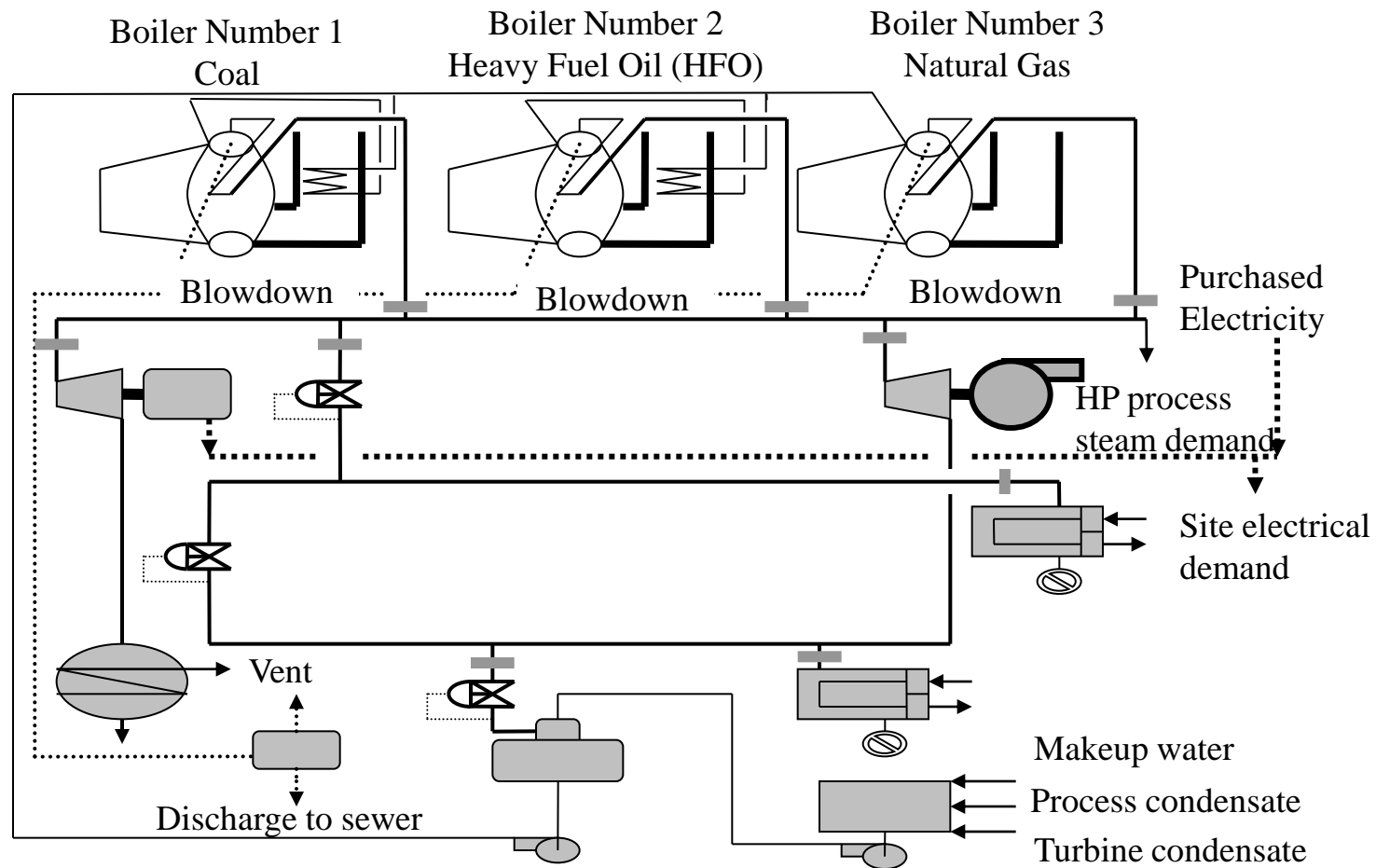
Modeling Backpressure Turbines in SSAT

Hands-On Student Exercise

Condensing Turbines

Modeling Condensing Turbines in SSAT

Steam System

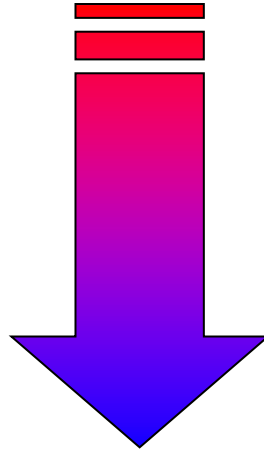


+ Indicates a flow meter installation

Turbines 101

- What is a Turbine?
 - Energy Conversion Device

Potential / Kinetic / Pressure / Thermal Energy



Rotational Shaft Energy

Users of Steam Turbines in Industry

➤ Heavy Steam Turbine Users

- Petrochemicals
- Petroleum Refining
- Forest Products (Pulp & Paper)
- Rubber
- Pharmaceuticals
- Manufacturing Assembly

➤ Medium & Small Steam Turbine Users

- Food & Beverage
- Plastics
- Electronics
- Metal Fabrication

Users of Steam Turbines in Industry

➤ Steam turbine drives commonly used in industry

- Direct power generation
- Boiler feed water pumps
- Cooling tower water pumps
- Chilled water pumps
- Boiler forced draft fans
- Exhaust fans
- Air compressors
- Refrigeration machines
- Chiller systems
- Other utility services



Users of Steam Turbines in Industry

➤ Steam Turbines

- Many different kinds
 - Backpressure
 - Condensing
 - Extraction
 - Combination
- Different size and efficiency ranges
- Backpressure turbines are used in lieu of letdown stations and in parallel with letdown stations
- Condensing turbines provide maximum shaft power per unit of steam flow



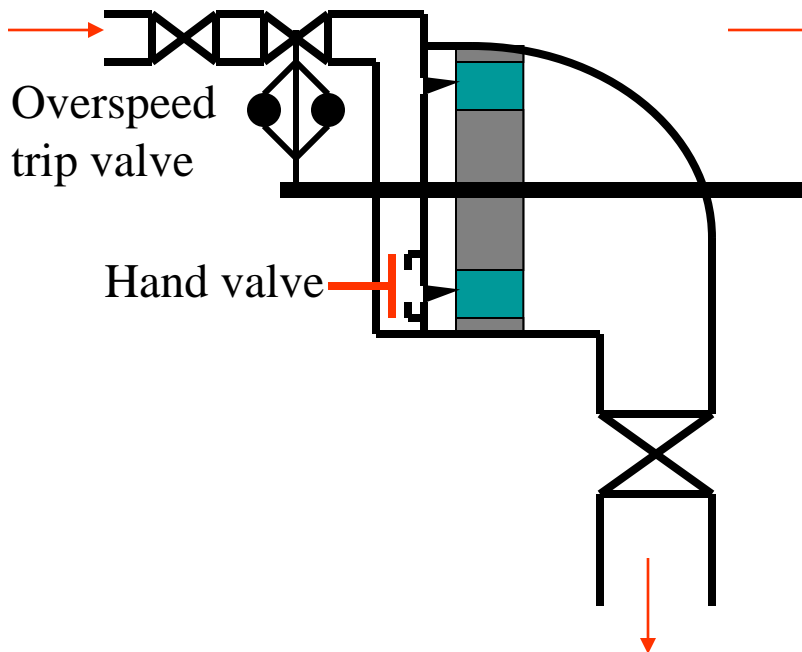
Backpressure Steam Turbines

- Backpressure steam turbines discharge steam at a pressure greater than (or equal to) atmospheric pressure

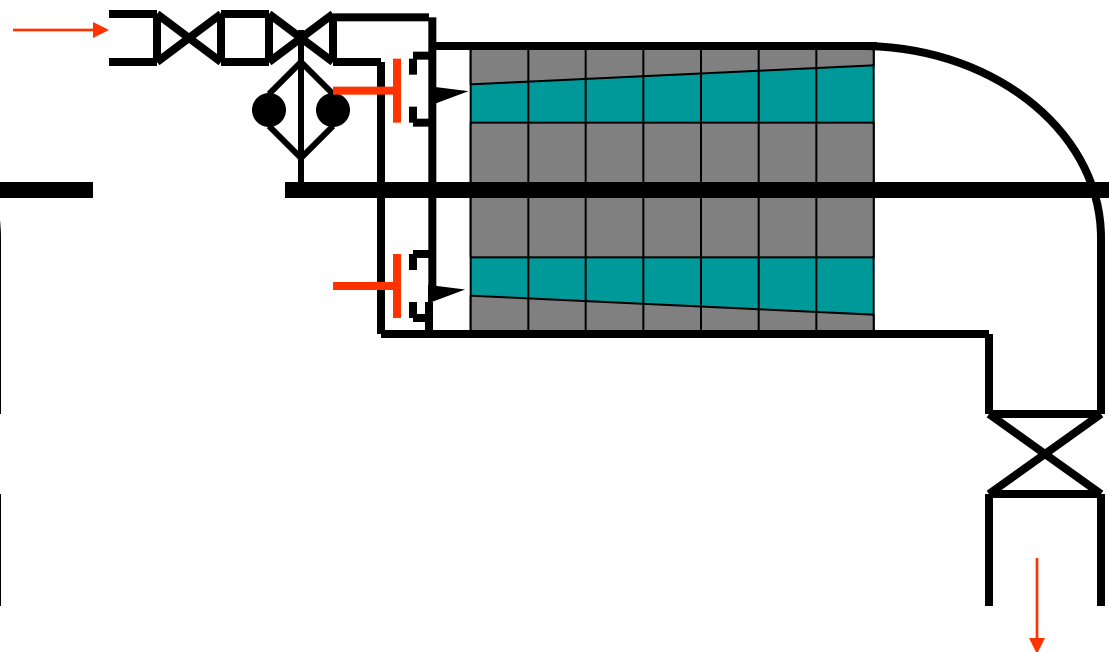
Stop valve

Overspeed
trip valve

Hand valve



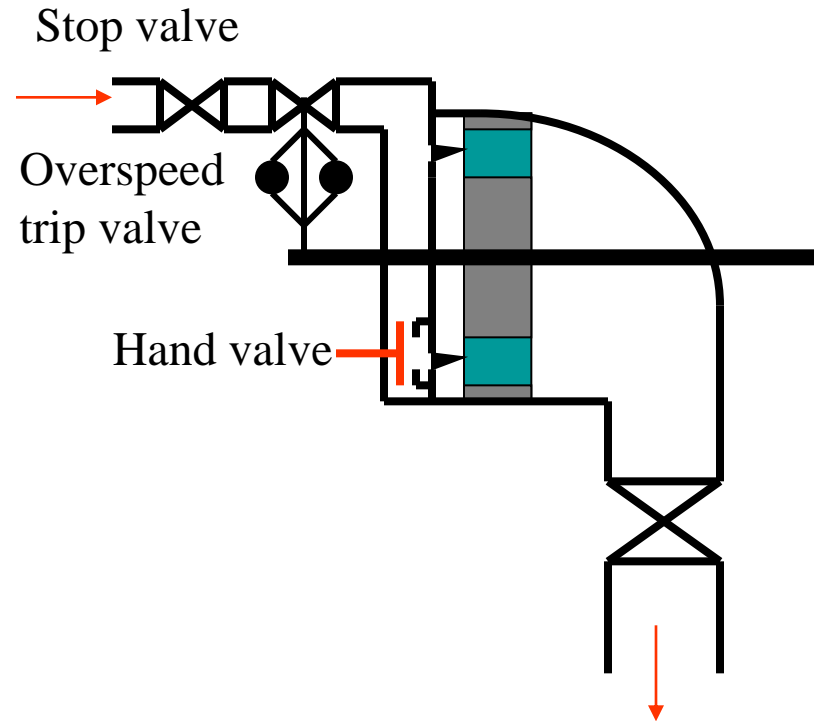
Single Stage Backpressure Turbine



Multistage Backpressure Turbine

Backpressure Steam Turbines

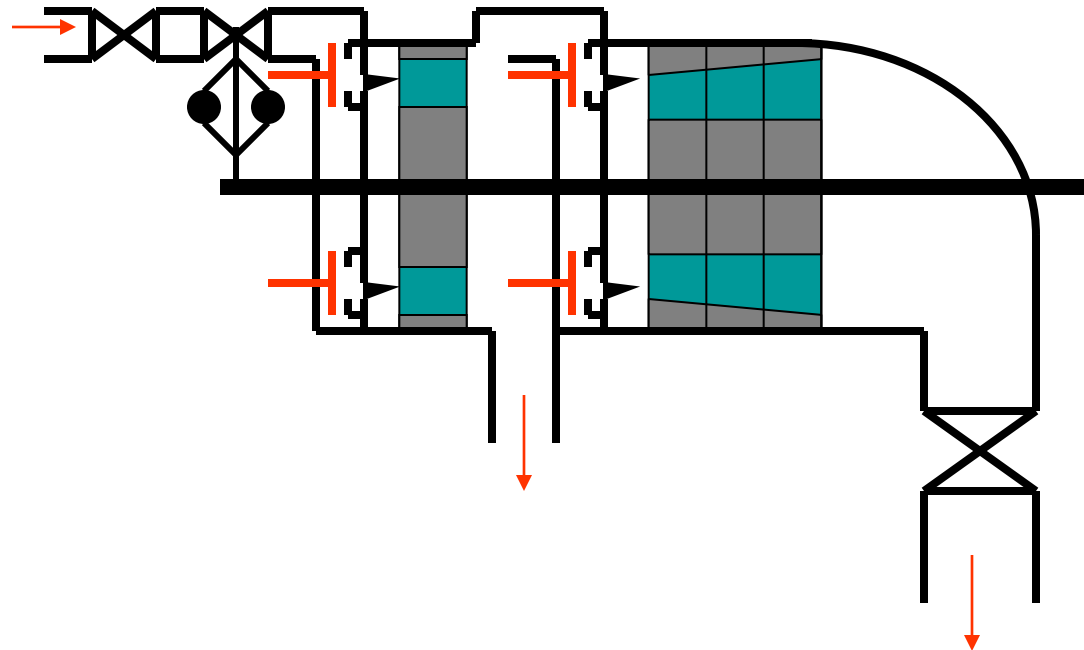
- Very common
- Simplest form
- Works against a backpressure
- Exhausts to a process load or steam header
- An excellent candidate for industrial applications



Single Stage Backpressure Turbine

Extraction Steam Turbines

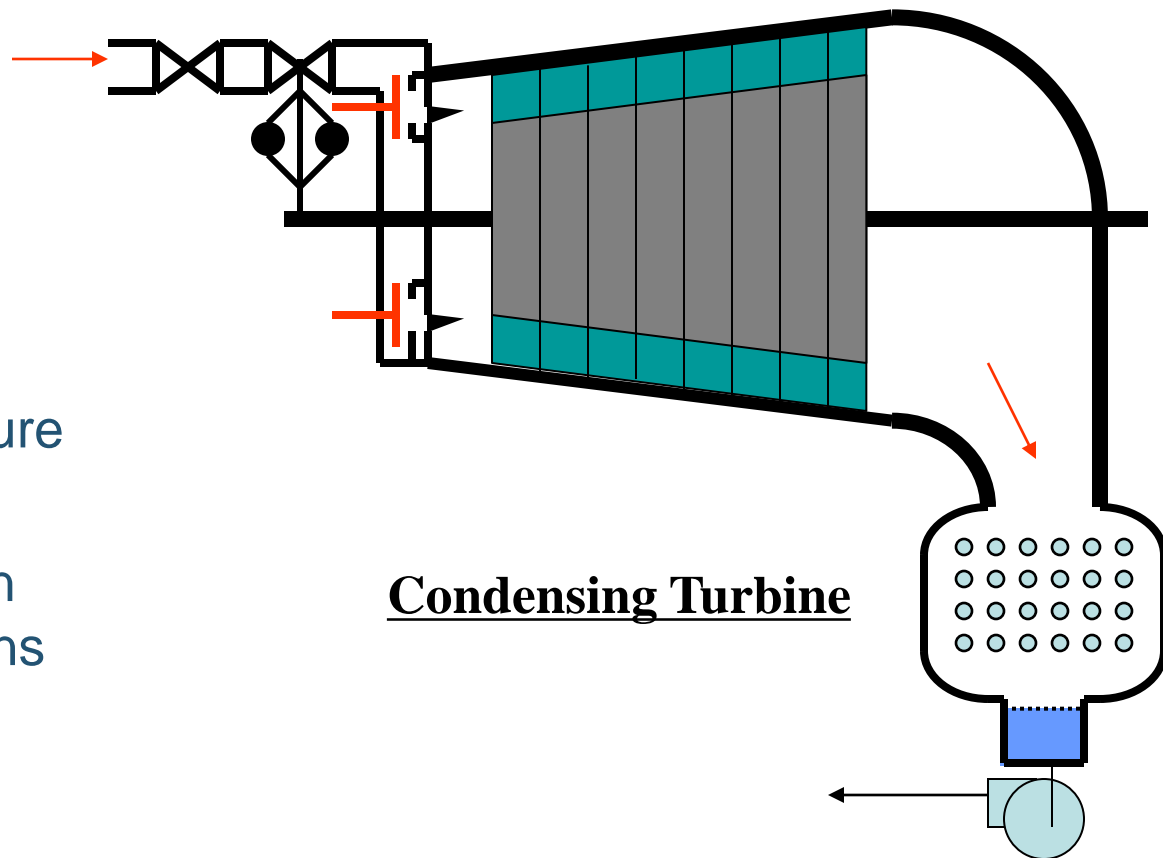
- Very common in plants that have multiple steam pressure headers
- Works against a backpressure
- Exhausts to a process load, steam header or a condenser
- An excellent candidate for balancing headers & eliminating steam venting



Extraction Turbine

Condensing Steam Turbines

- The industry workhorse for power generation
- Will always have an associated steam condenser
- Exhausts to vacuum
- Highest operating pressure ratios
- Multistage and may even have two or three sections
- Very large sizes
- Lowest steam rates



Typical Industrial Steam Turbines Operations

- Operating pressures
 - Minimum – 10 bars (for backpressure)
 - Maximum – 100 bars
 - Vacuum conditions can exist at the exhaust!
- Operating steam temperatures
 - Saturated or a few degrees of superheat
 - Maximum – 200°C superheat
- Summary – Steam turbine technology is very diverse and operates over a broad range of pressures and temperatures

Turbine First Law Efficiency

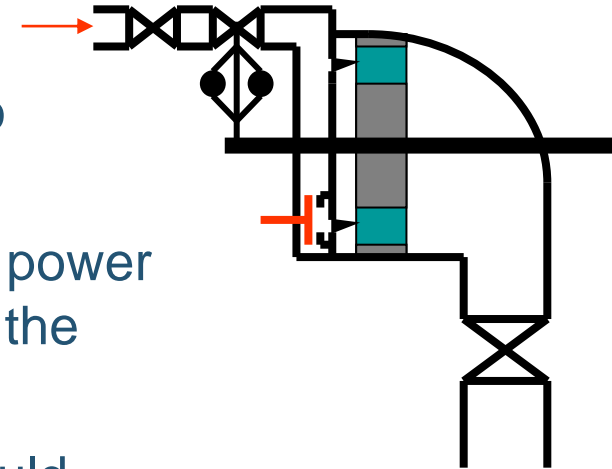
- An energy balance conducted on a steam turbine will reveal an exceptionally high efficiency
 - Essentially all of the energy taken out of the steam is converted into shaft energy

$$\eta_{first\ law} = \frac{\dot{W}_{shaft}}{\dot{m}_{steam}(h_i - h_e)} \approx 100\%$$

- Steam turbines operate with only minor “losses”
 - Bearing friction
 - Heat transfer
 - Gland losses

The Perfect Turbine

- Steam turbines are evaluated using the *Second Law of Thermodynamics*
 - The Second Law of Thermodynamics identifies that thermal energy cannot be converted completely into power
 - Power can be converted completely into thermal energy
 - This defines the maximum amount of shaft power that could possibly be produced (based on the laws of physics)
 - This defines a *perfect turbine*, which would operate *isentropically*
 - *Isentropic is constant entropy (no losses)*
 - » No entropy generation



Isentropic Efficiency

- Steam turbine efficiency is described as *isentropic efficiency*
 - A comparison of the actual work produced compared to a perfect (isentropic) turbine

$$\eta_{isentropic} = \frac{\text{Actual Work}}{\text{Isentropic Work}} = \frac{\dot{W}_{actual}}{\dot{W}_{isentropic}}$$

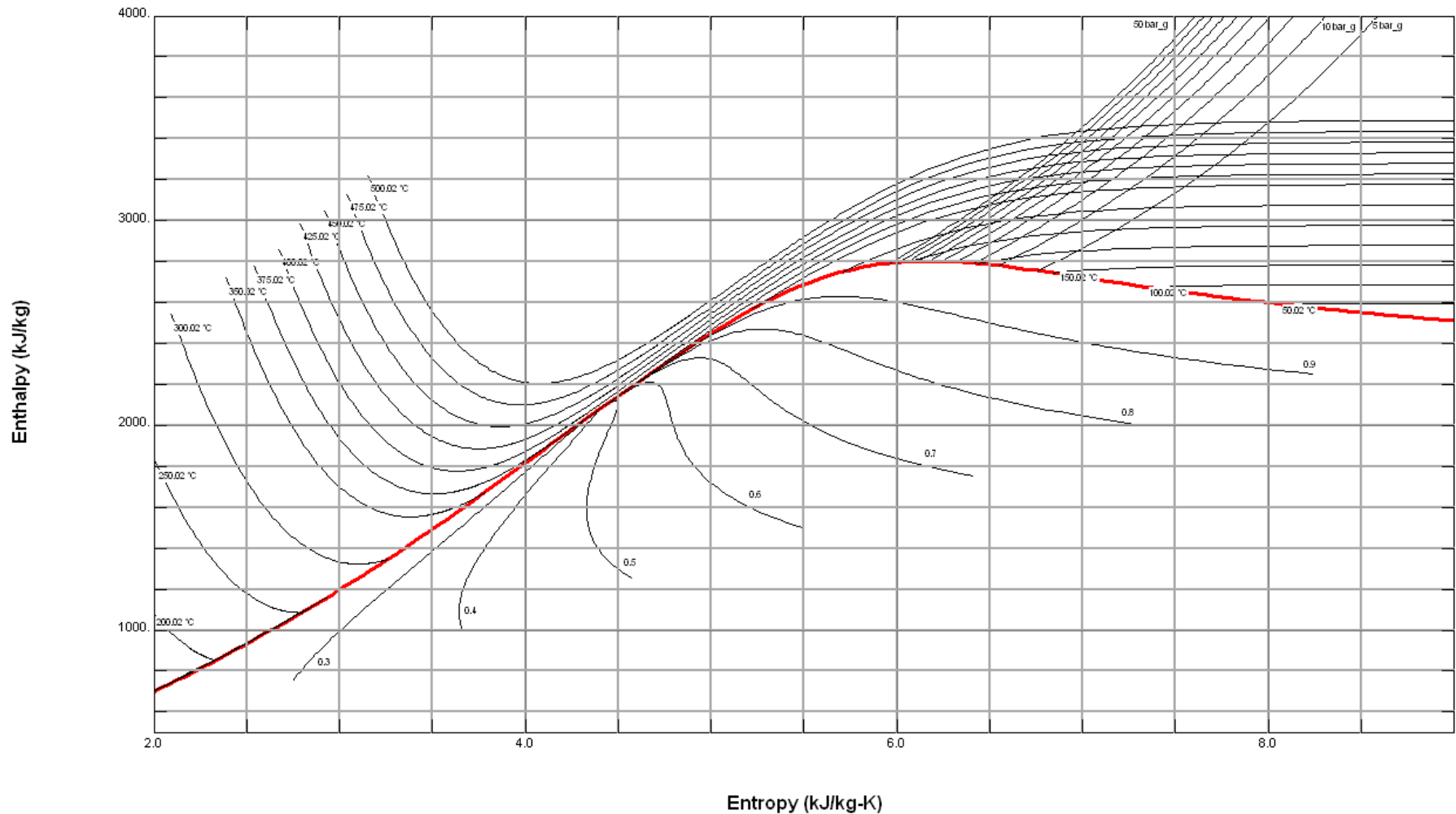
$$\eta_{isentropic} = \frac{\dot{m}_{steam}(h_{inlet} - h_{exit})_{actual}}{\dot{m}_{steam}(h_{inlet} - h_{exit})_{isentropic}} = \frac{(h_i - h_e)_{actual}}{(h_i - h_e)_{isentropic}}$$

Isentropic Efficiency

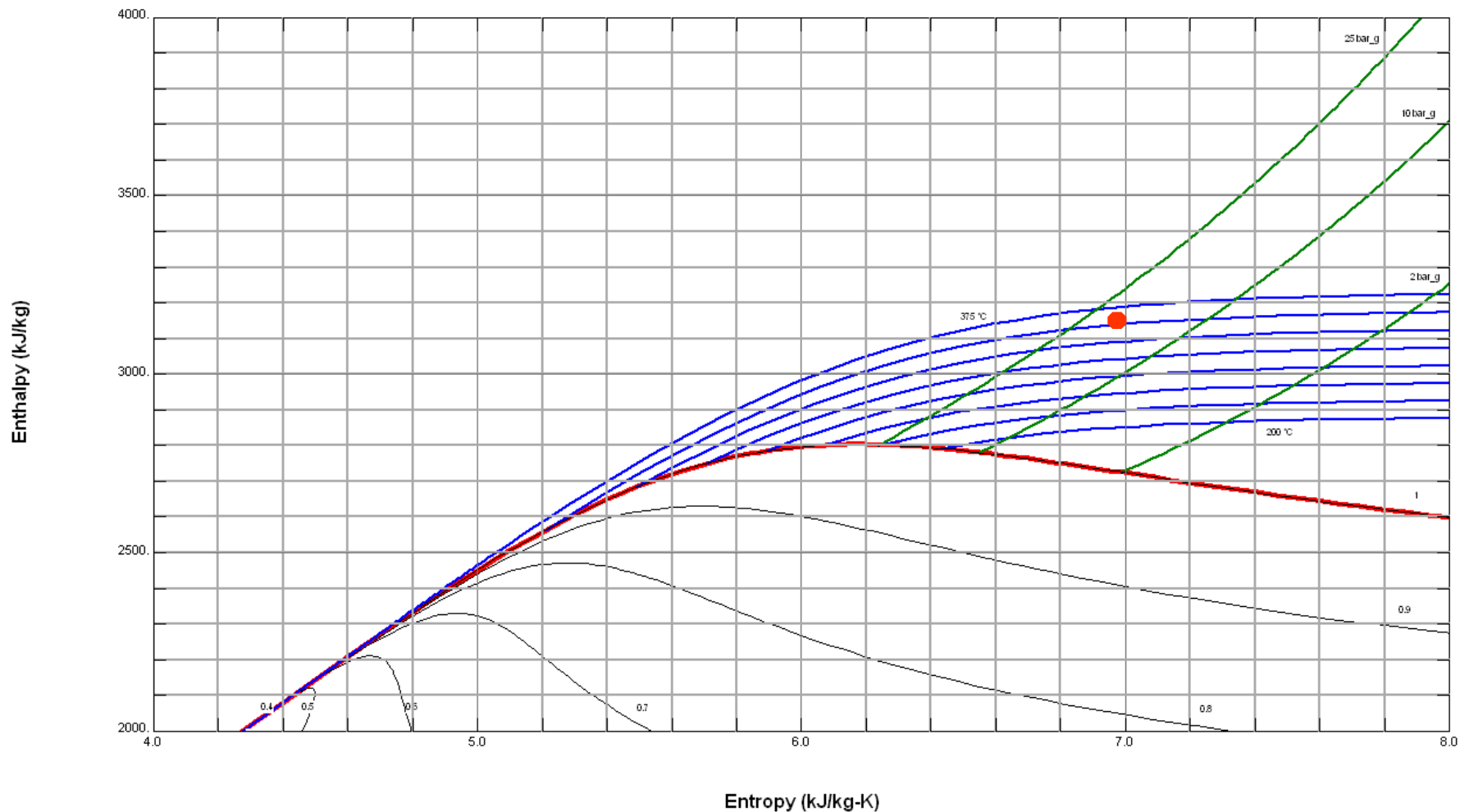
- Steam turbine efficiency is not “like” boiler efficiency
 - Turbine isentropic efficiency is a comparison of the actual turbine operation to that of a perfect turbine operating with the same inlet conditions and outlet pressure
 - Isentropic efficiency is a description of how much mechanical energy is developed from thermal energy

- The steam exiting the turbine contains a significant amount of useful thermal energy

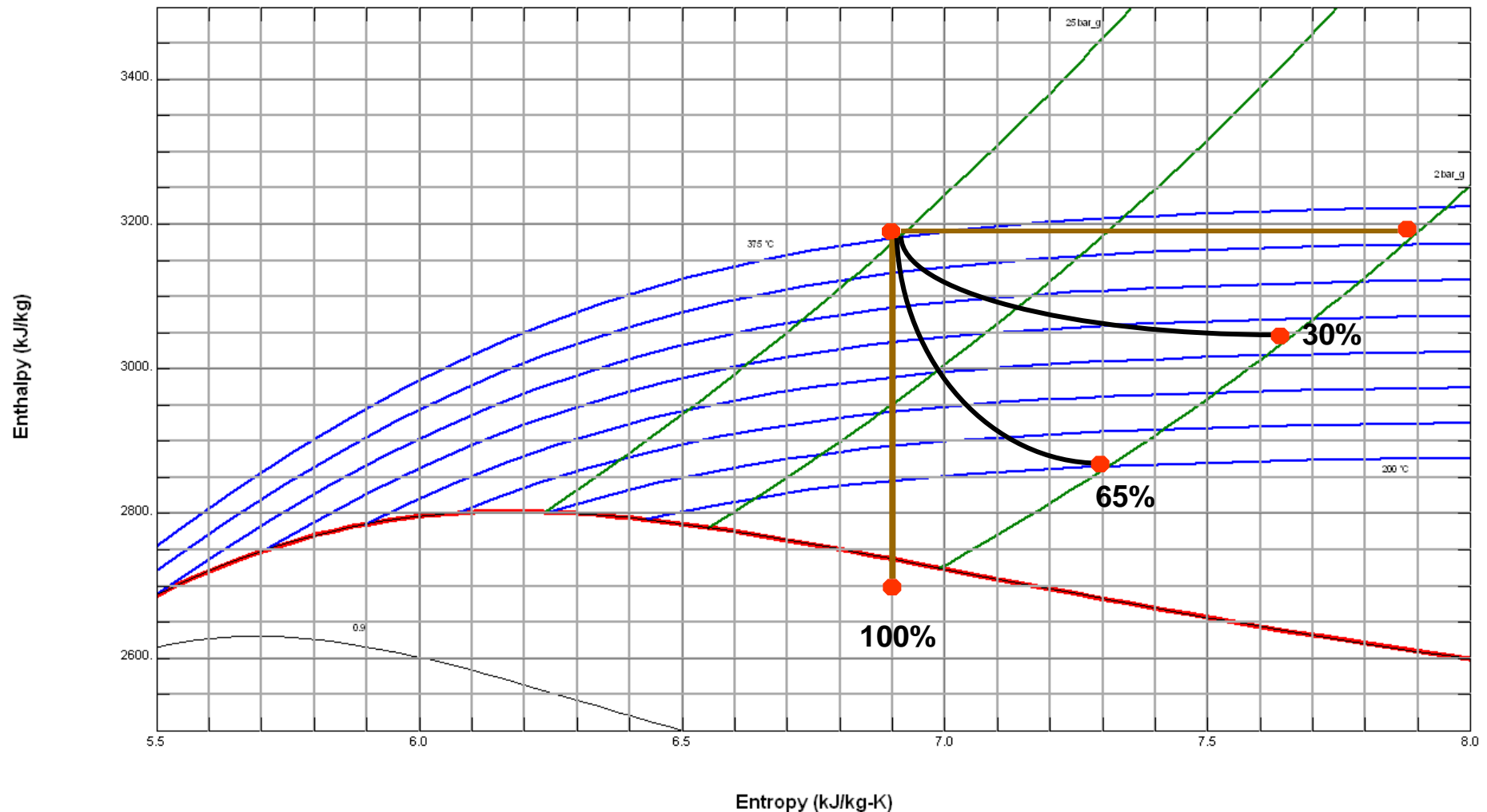
Mollier Diagram



Mollier Diagram



Mollier Diagram – Isentropic Turbine Efficiency



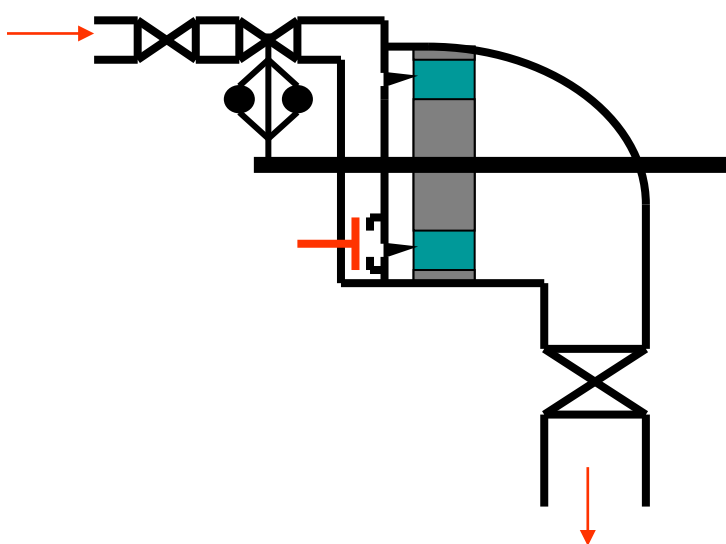
Typical Steam Turbine Efficiency

$$\eta_{isentropic} = \frac{(h_{in} - h_{out})_{actual}}{(h_{in} - h_{out})_{isentropic}} = 20\% \text{ to } 80\%$$

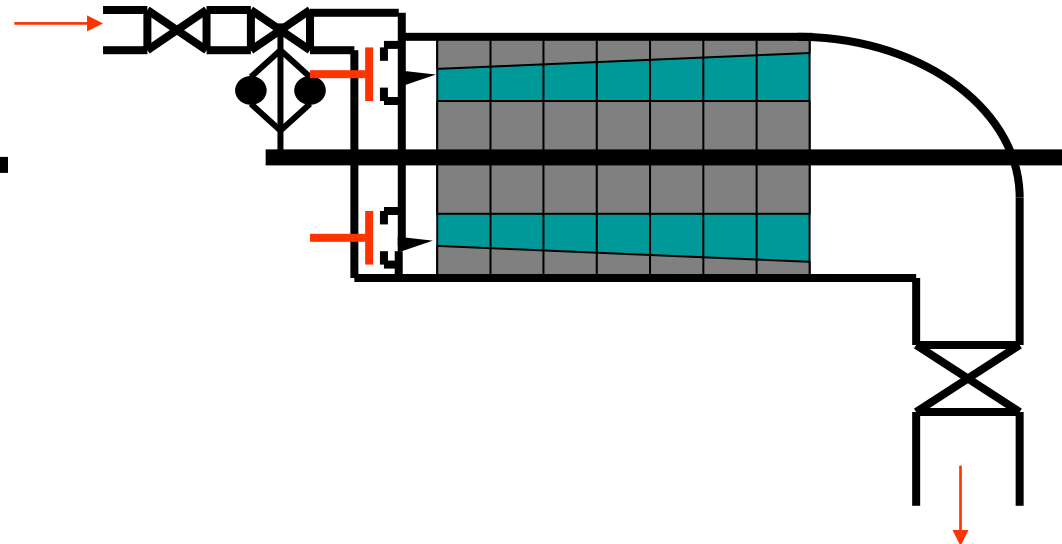
- Major contributors to isentropic efficiency
 - Turbine design
 - Control valve type
 - Single valve – throttle
 - Multi-valve – flow nozzles
- Will need this information for ANY turbine analysis

Steam Turbine Efficiency

- Generally, single stage turbines operate with lower isentropic efficiency than multistage turbines
 - Increasing steam path area (diameter) decreases losses
 - Steam exhaust velocity and wall friction decrease
 - Single stage turbines are typically more efficient than multistage turbines for small capacity machines



Single Stage Backpressure Turbine



Multistage Backpressure Turbine

Steam Turbine Efficiency

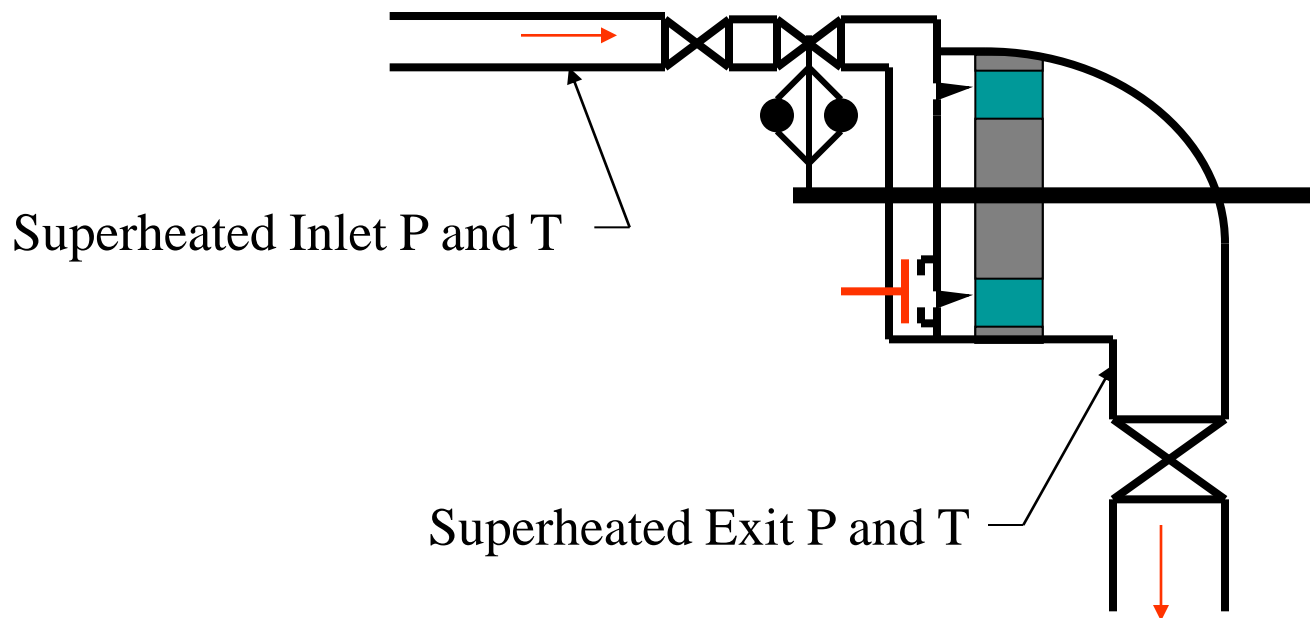
- Three methods of obtaining isentropic turbine efficiency:
 - Manufacturer specifications
 - Turbine maps / performance curves
 - Excellent starting point – for new designs also
 - Steam inlet and outlet conditions known
 - Superheated inlet along with superheated outlet is the most common and easiest to utilize
 - Will NOT work with saturated outlets (quality < 1)
 - Steam inlet conditions and power generation known
 - Typically used for electrical power generation units
 - Mass flow rate of steam will be required
 - Alternate option may exist for direct mechanical driven equipment but with higher uncertainty

Turbine Efficiency from Inlet and Outlet Steam Conditions

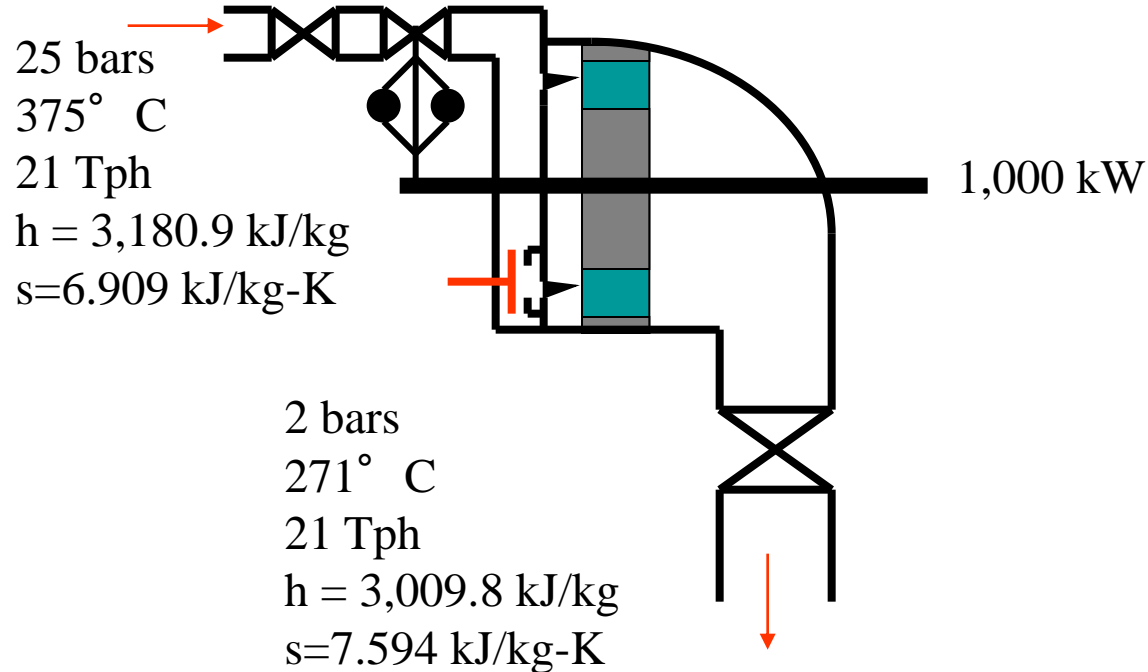
- Method 2
- Turbine performance can be determined from inlet and outlet steam conditions and steam properties associated at those conditions

Steam Turbine Efficiency

- For superheated steam conditions at the turbine inlet and outlet
 - Pressure and temperature measurements of superheated steam allow all of the thermodynamic properties to be known



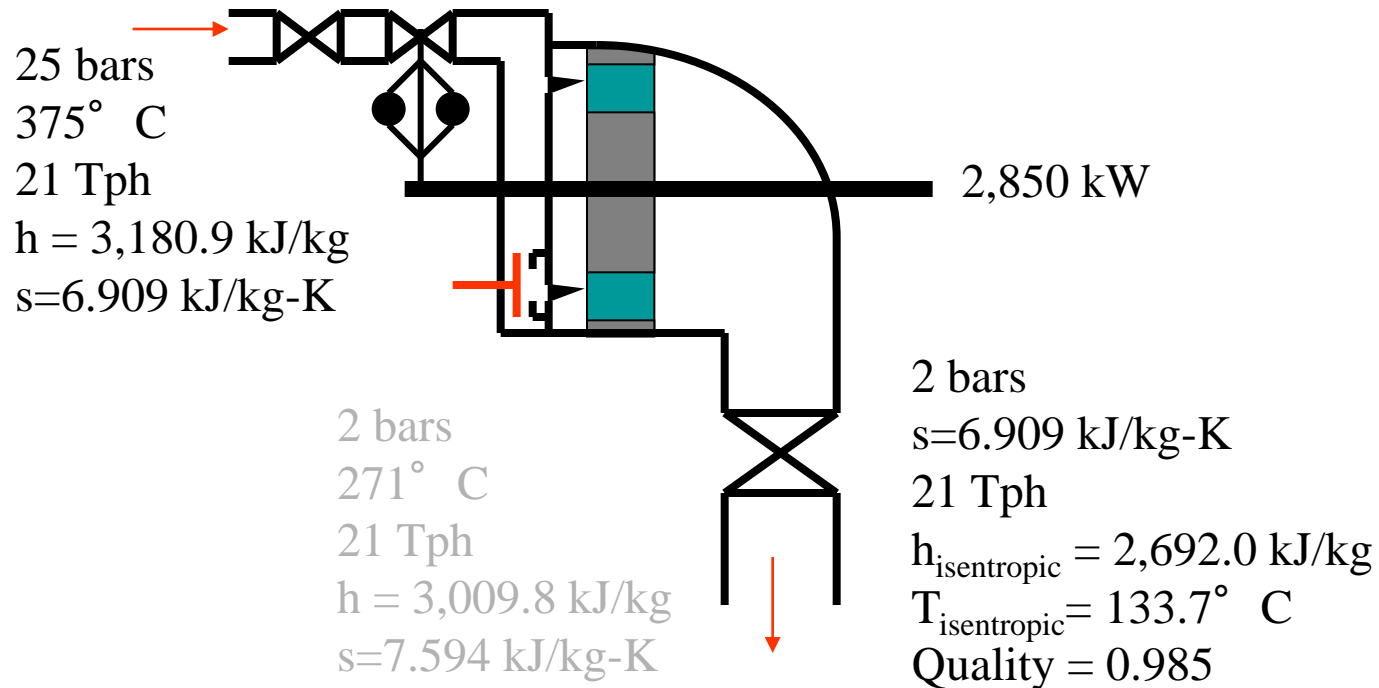
Actual Operating Conditions



$$\dot{W}_{shaft} = \dot{m}_{steam} (h_i - h_e)_{steam} = \frac{21,000}{3,600} (3,180.9 - 3,009.8)$$

$$\dot{W}_{shaft} = 1,000 \text{ kW} = 1,000 \text{ kW} \left(\frac{1 \text{ hp}}{0.746 \text{ kW}} \right) = 1,340 \text{ hp}$$

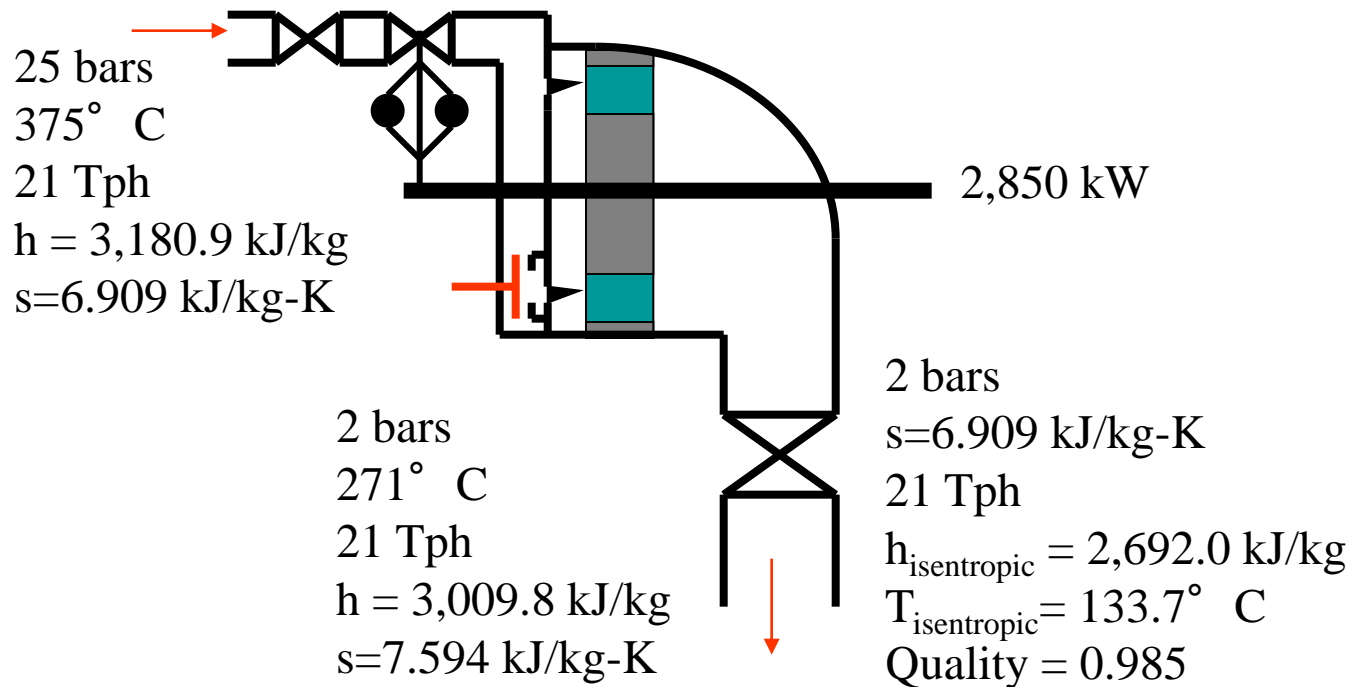
Isentropic Conditions



$$\dot{W}_{\text{shaft}} = \dot{m}_{\text{steam}} (h_i - h_e)_{\text{steam}} = \frac{21,000}{3,600} (3,180.9 - 2,692.0)$$

$$\dot{W}_{\text{shaft}} = 2,850 \text{ kW} = 2,850 \text{ kW} \left(\frac{1 \text{ hp}}{0.746 \text{ kW}} \right) = 3,825 \text{ hp}$$

Isentropic Conditions



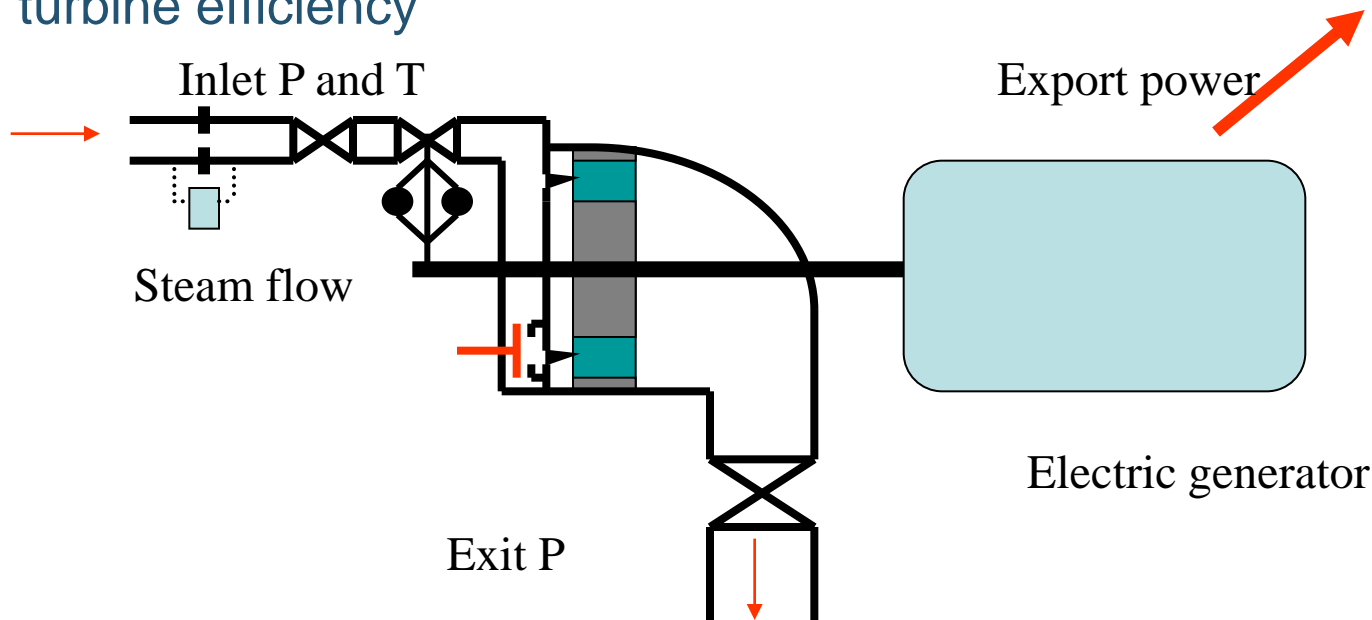
$$\eta_{\text{isentropic}} = \frac{(h_{\text{inlet}} - h_{\text{exit}})_{\text{actual}}}{(h_{\text{inlet}} - h_{\text{exit}})_{\text{isentropic}}} = \frac{(3,180.9 - 3,009.8)}{(3,180.9 - 2,692.0)} = \frac{171.1}{488.9} = 0.35$$

Turbine Efficiency from Steam Conditions and Power Production

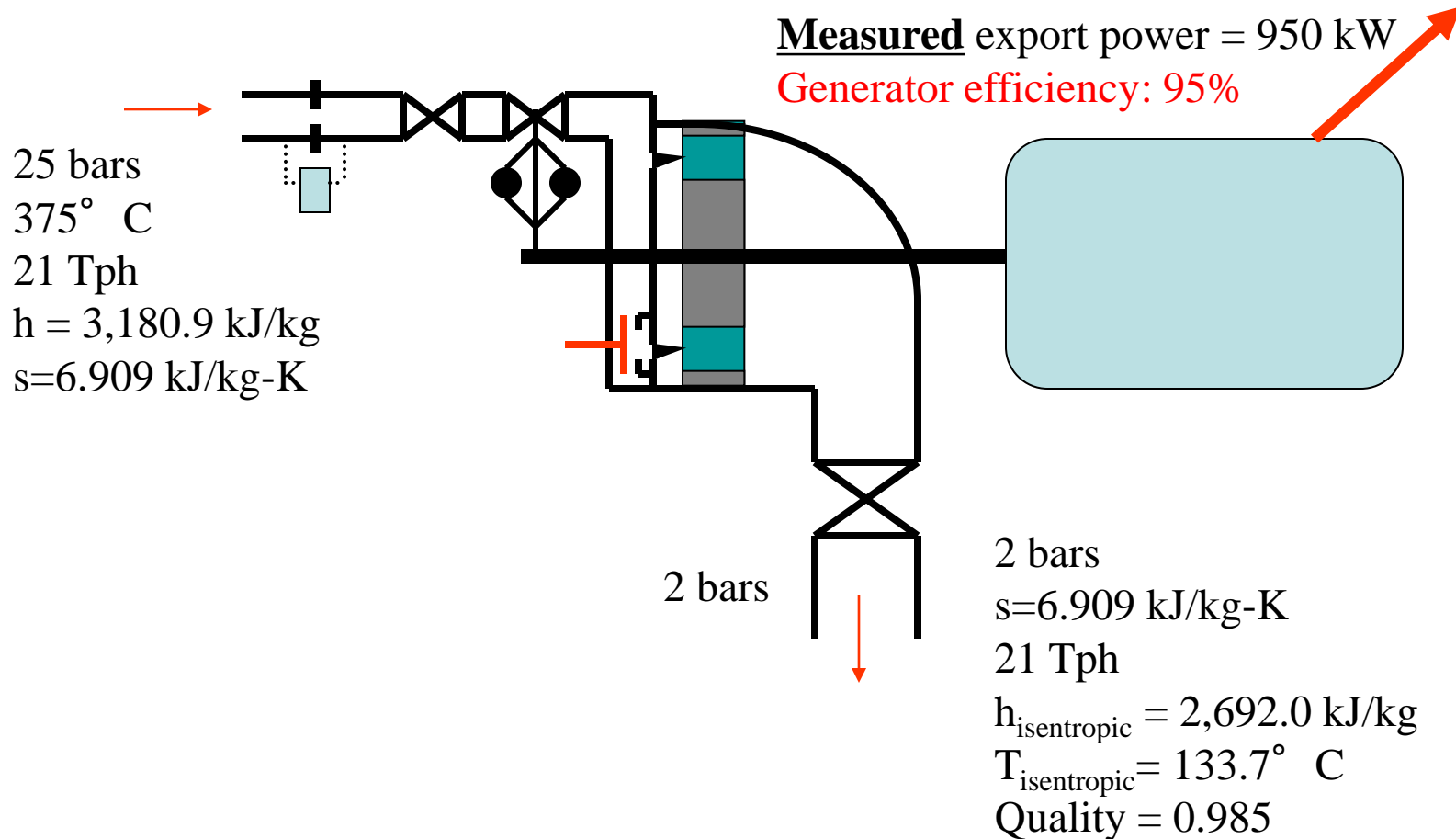
- Method 3
- Turbine performance can be determined from inlet steam properties and power production

Steam Turbine Generator Efficiency

- Steam turbines coupled with electric generators provide an additional mechanism for calculating turbine isentropic efficiency
 - Additional measurements are required to allow the efficiency determination
 - This is typically the only practical method to evaluate condensing turbine efficiency



Steam Turbine Efficiency



Steam Turbine Efficiency

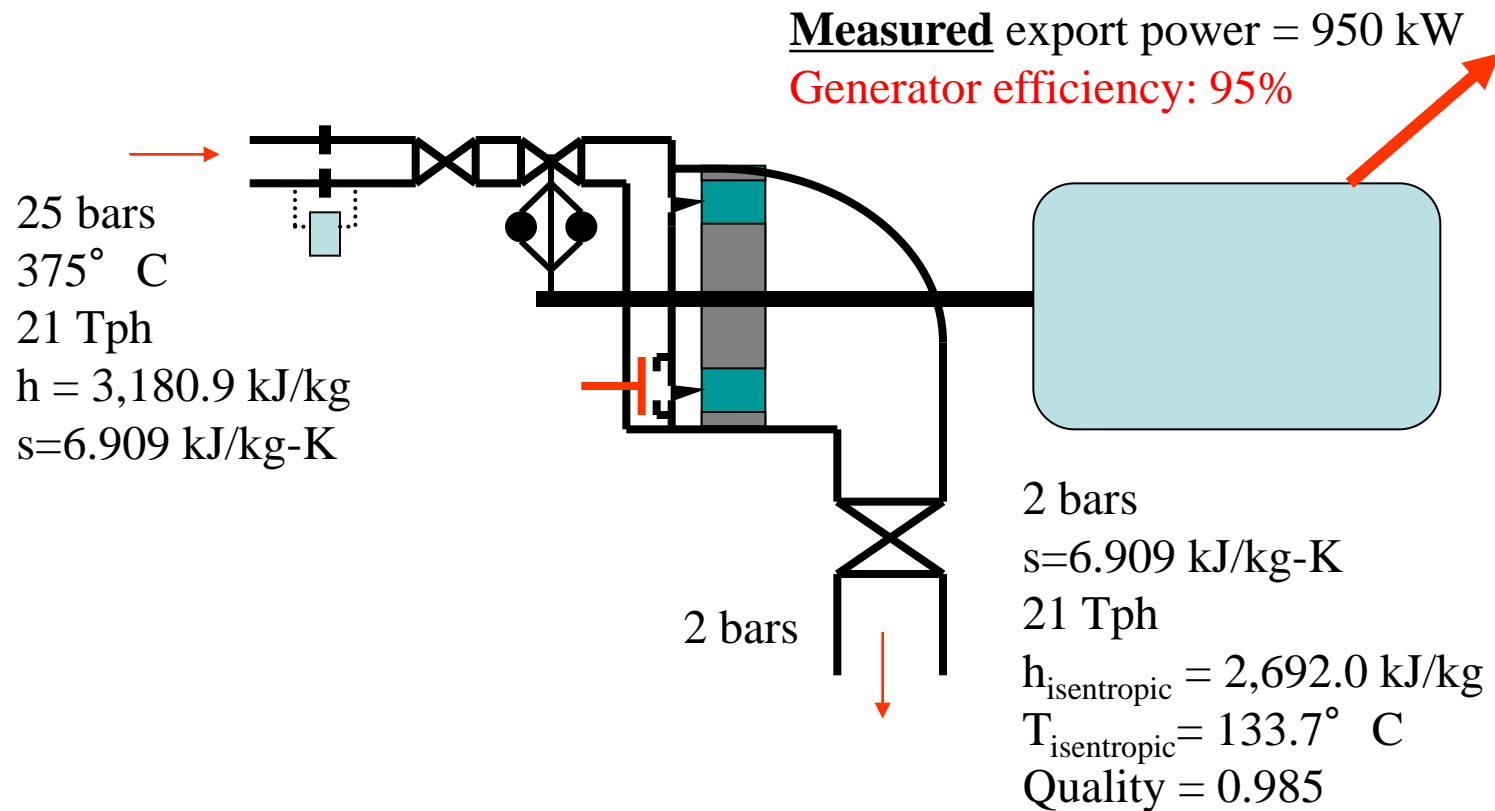
$$\dot{W}_{generator} = 950 \text{ kW}$$

$$\eta_{generator} = \frac{\text{Generator Work}}{\text{Turbine Shaft Work}} = \frac{\dot{W}_{generator}}{\dot{W}_{turbine}} = \frac{950 \text{ kW}}{\dot{W}_{turbine}} = 0.95$$

$$\dot{W}_{turbine} = 1,000 \text{ kW}$$

$$\eta_{isentropic} = \frac{\text{Actual Turbine Work}}{\text{Isentropic Work}} = \frac{\dot{W}_{generator}}{\eta_{generator} \dot{W}_{isen}} = \frac{\dot{W}_{gen}}{\eta_{gen} \dot{m}_{st} (h_i - h_e)_{isen}}$$

Steam Turbine Efficiency



$$\eta_{\text{isentropic}} = \frac{950 \text{ kW}}{0.95} \frac{3,600 \frac{\text{s}}{\text{hr}}}{21,000 \frac{\text{kg}}{\text{hr}}} \frac{1}{\left(3,180.9 \frac{\text{kJ}}{\text{kg}} - 2,692.0 \frac{\text{kJ}}{\text{kg}}\right)} = 0.35$$

Steam Rate

- *Steam rate* is an expression used to describe the amount of steam required to produce a specific amount of power
 - *Theoretical steam rate* is the ideal steam rate
 - *Actual steam rate* is the real world steam rate

$$\text{Theoretical Steam Rate} = \text{TSR} = \frac{\dot{m}_{\text{steam}}}{\dot{W}_{\text{isentropic}}} = \frac{1}{(h_1 - h_{2\text{isen}})}$$

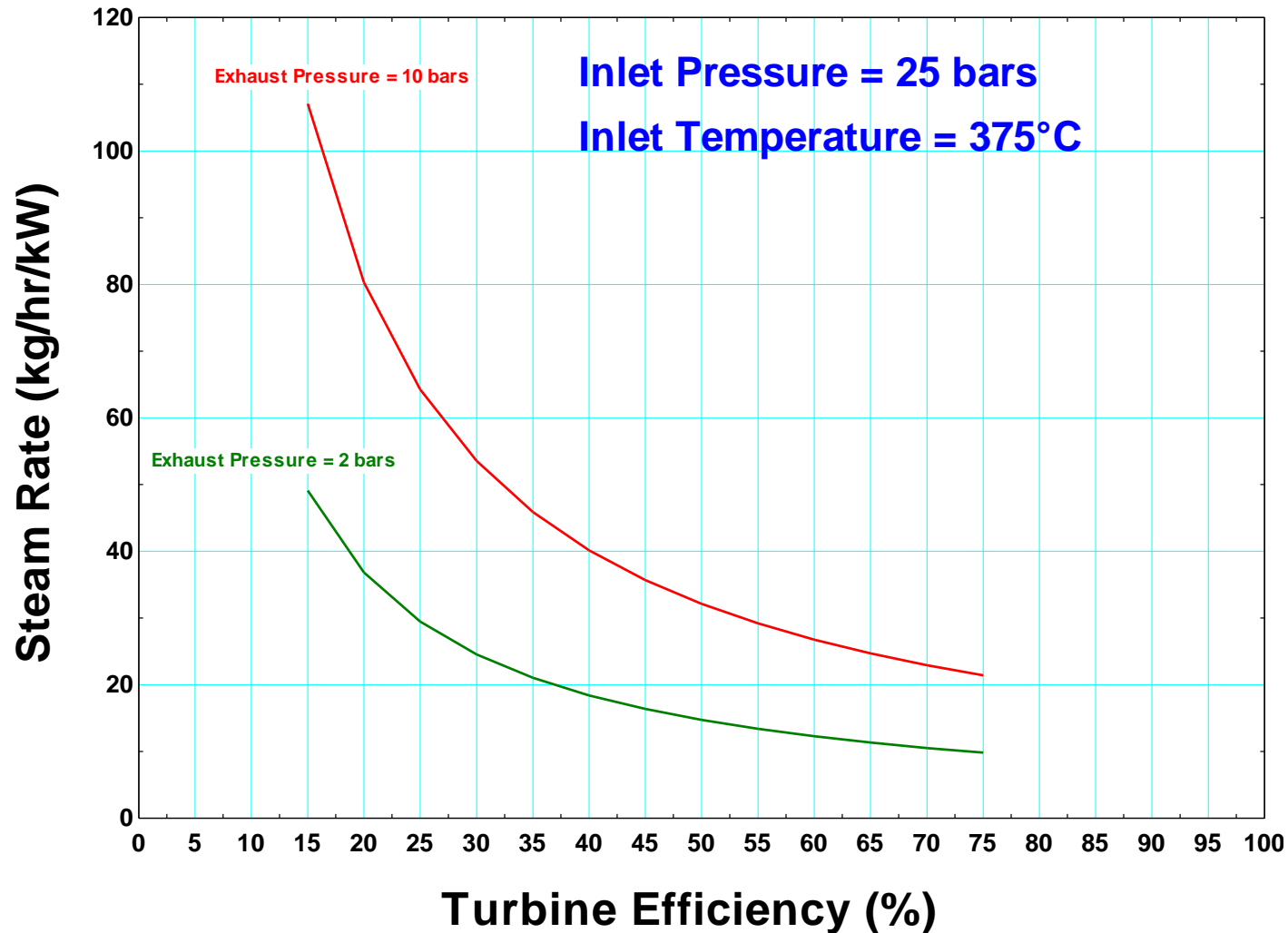
$$\text{Actual Steam Rate} = \text{ASR} = \frac{\dot{m}_{\text{steam}}}{\dot{W}_{\text{actual}}} = \frac{1}{(h_1 - h_2)}$$

$$\eta_{\text{isen}} = \frac{\text{TSR}}{\text{ASR}}$$

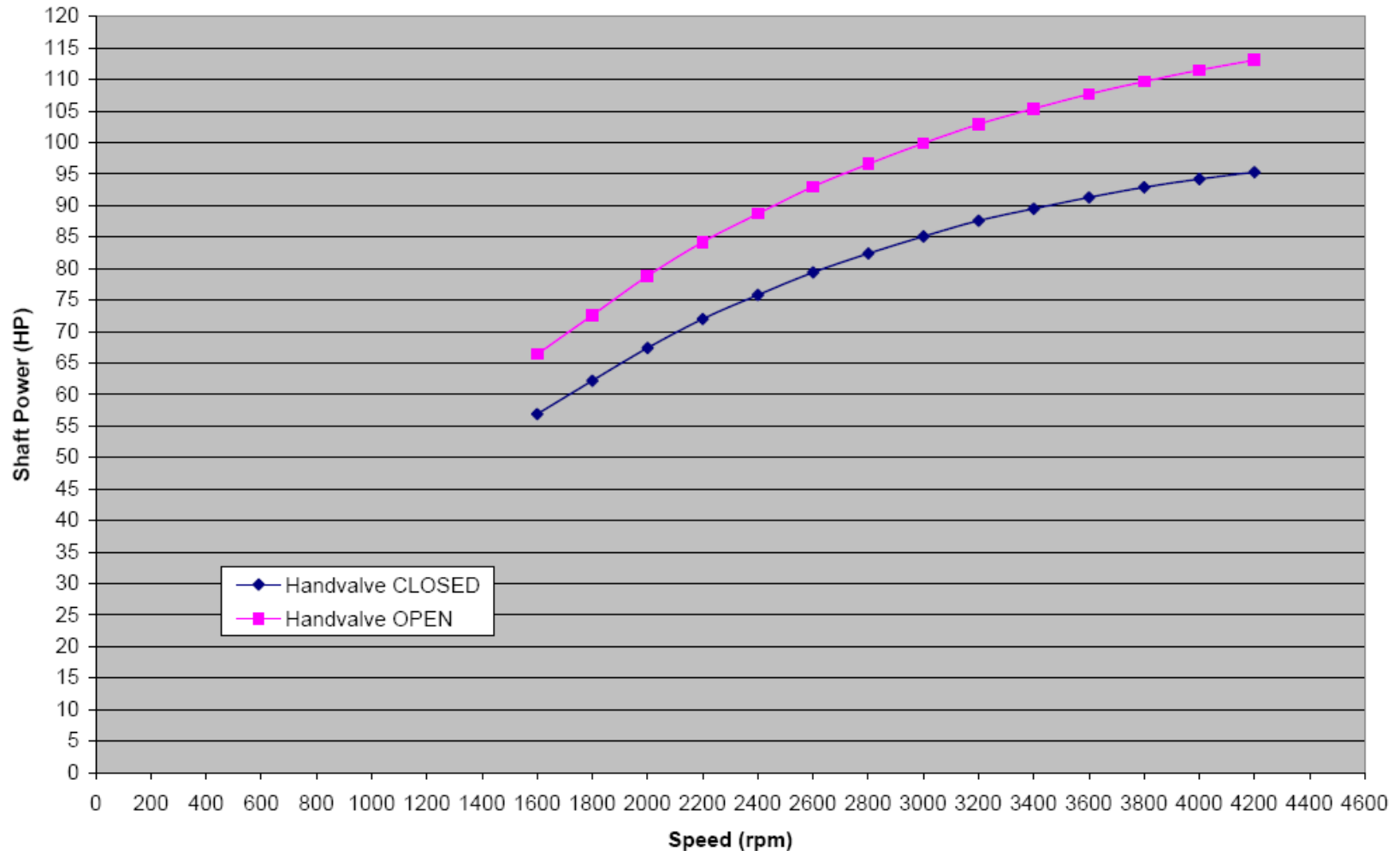
Steam Rate Factors

- Changing turbine inlet or outlet conditions will not impact isentropic efficiency significantly
 - Steam rate will change dramatically if conditions are changed
- Throttling the inlet of a steam turbine will impact the overall isentropic efficiency (valve inlet to turbine outlet)
 - The isentropic efficiency of the turbine will not change significantly (turbine inlet to outlet)
 - Steam rate will change dramatically

Steam Rate & Efficiency



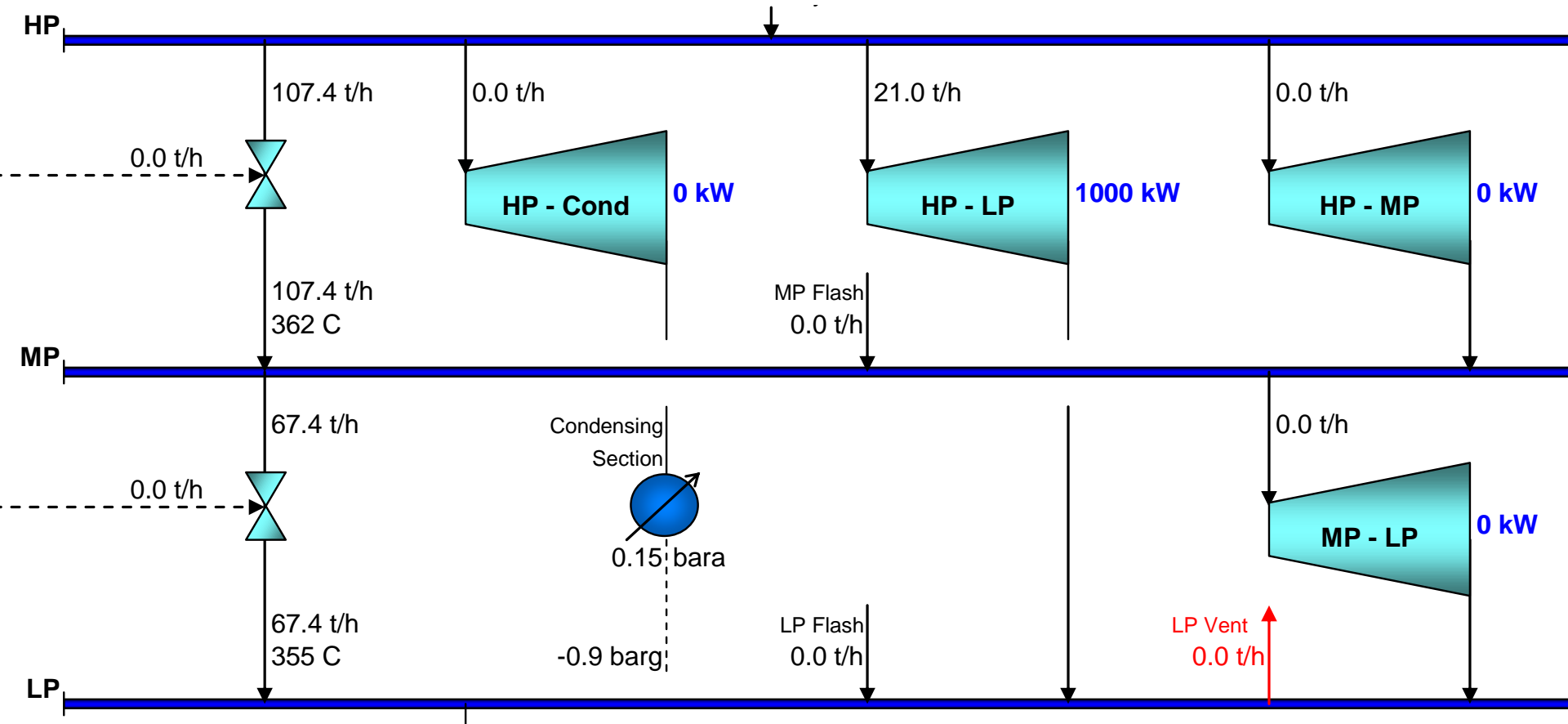
Steam Turbine Speed



SSAT Steam Turbine Applications

- Total Site Electrical Demand is held constant in SSAT project evaluations
- Power produced by the turbines reduces the Power Import
- SSAT incorporates a maximum of only four turbines
 - HP – LP
 - HP – MP
 - MP – LP
 - HP – Condensing
 - The Impact Turbine must be modeled
- *Site Detail* section allows actual performance to be incorporated into the analysis
 - Turbine efficiency
 - Pressure reducing valve interaction
 - Turbine capacity and control

Steam Turbines Schematically in SSAT



Steam Turbines in SSAT

Steam Turbines		
Do you have a steam turbine installed between HP and LP?	Yes	▼
Do you have a steam turbine installed between HP and MP?	No	▼
Do you have a steam turbine installed between MP and LP?	No	▼
Do you have an HP to condensing turbine installed?	No	▼

- Steam turbines need to be set up in the “Quick Start” section
 - Use the pull down menu to say “Yes” if a particular turbine exists
- Additional information on turbines and their control mechanism is then provided in the “Site Detail” section
- NOTE: The turbine does not have to be ON for inclusion in the system

Steam Turbines in SSAT

HP to LP Steam Turbine(s)	Input Data	Notes/Warnings
→ Isentropic efficiency	35 %	

Note: If multiple turbines are installed, the operation of the impact turbine (the turbine affected by changes to the system) should be modeled

Note: A generator electrical efficiency of 100% is assumed by the model

→ Select the appropriate turbine operating mode	Option 2 - Fixed operation	▼
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Note: If Option 1 is chosen, the model will preferentially use the HP to LP turbine to balance the LP demand

→ Option 2 - How should the fixed turbine operation be defined?	Specify fixed power generation	▼
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Option 2 - Fixed steam flow	50 t/h	
→ Option 2 - Fixed power generation	1000 kW	

Option 3 - How do you wish to define the operating range?	Option 3 not selected	▼
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Option 3 - Minimum steam flow	25 t/h	
Option 3 - Maximum steam flow	75 t/h	
Option 3 - Minimum power generation	1500 kW	
Option 3 - Maximum power generation	2500 kW	

- Each steam turbine included in the system will need to be configured for efficiency & control options

Steam Turbines in SSAT

HP to LP Steam Turbine(s)	Input Data	Notes/Warnings
→ Isentropic efficiency	35 %	←

Note: If multiple turbines are installed, the operation of the impact turbine (the turbine affected by changes to the system) should be modeled

Note: A generator electrical efficiency of 100% is assumed by the model

- Each steam turbine included in the system will need a steam turbine isentropic efficiency
 - Manufacturers' data
 - Calculated from steam input and output conditions for superheated cases
 - Calculated from power generated and steam input and outlet pressure
 - Generator efficiency needs to be included in the calculations
- Turbine operations are satisfied, then low-pressure demands are satisfied with PRV operation

Steam Turbines in SSAT

→ Select the appropriate turbine operating mode

Option 2 - Fixed operation



Note: If Option 1 is chosen, the model will preferentially use the HP to LP turbine to balance the LP demand

→ Option 2 - How should the fixed turbine operation be defined?

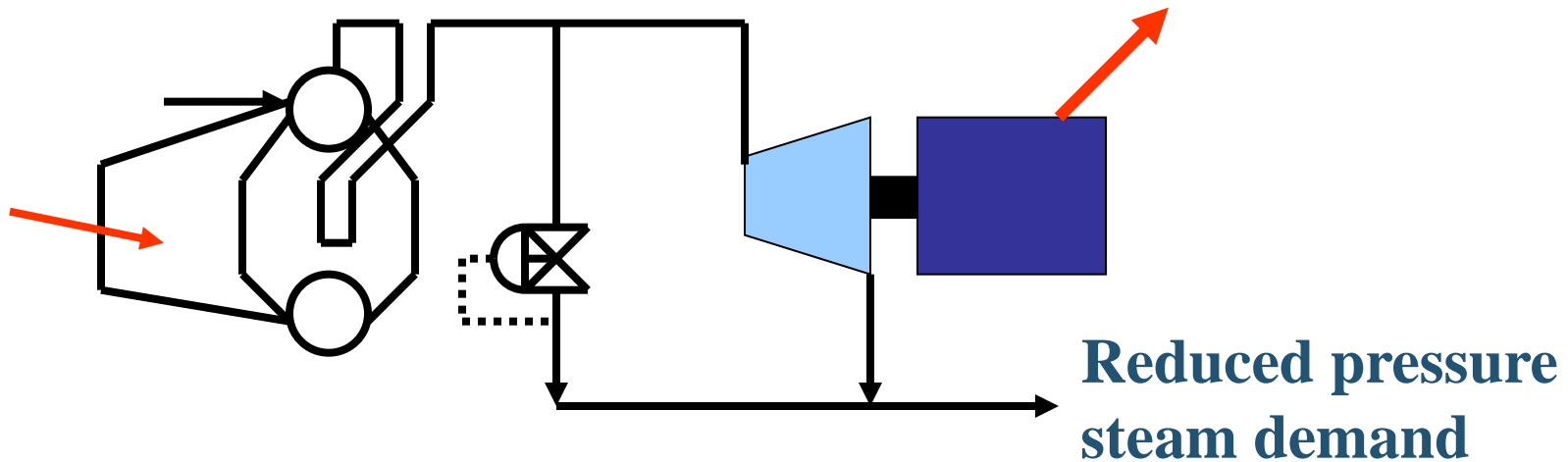
Specify fixed power generation



- There are four different options to setup the turbine operations
 - Steam flow to the turbine balances the “output” header demand
 - This is also the Default option
 - Turbine set up as fixed operation
 - Fixed steam flow
 - Fixed power generation
 - Turbine setup to operate between minimum and maximum limits
 - Steam flow
 - Power generation
 - Turbine NOT operating

Steam Balance Control

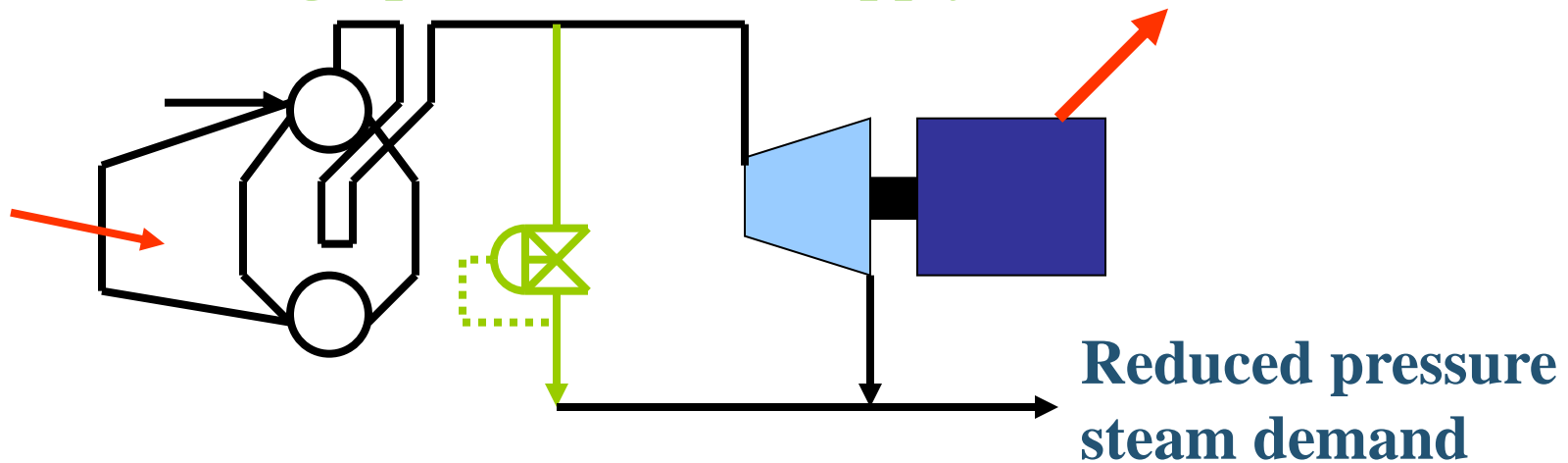
High-pressure steam supply



- The SSAT model must be established to accurately describe the **impact** a change in steam demand (or operating conditions) will have on the system
 - The **impact component** must be established
 - Steam can pass through a turbine
 - Steam can pass through a pressure reducing valve

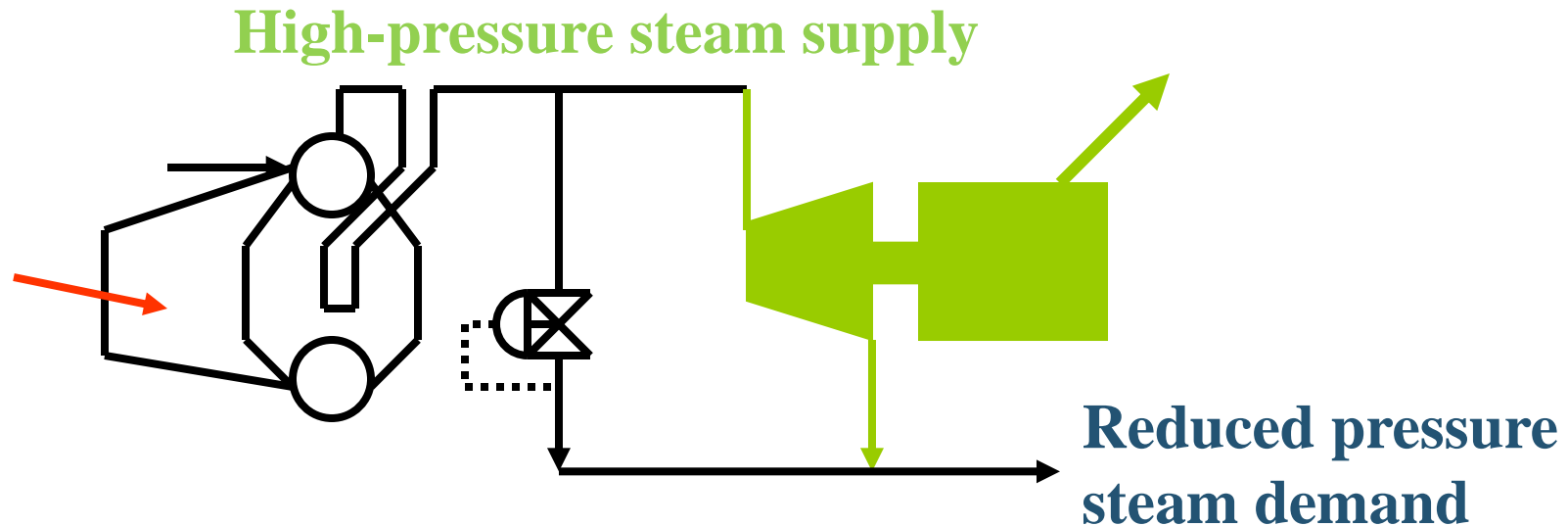
Turbine Control of Steam Balance

High-pressure steam supply



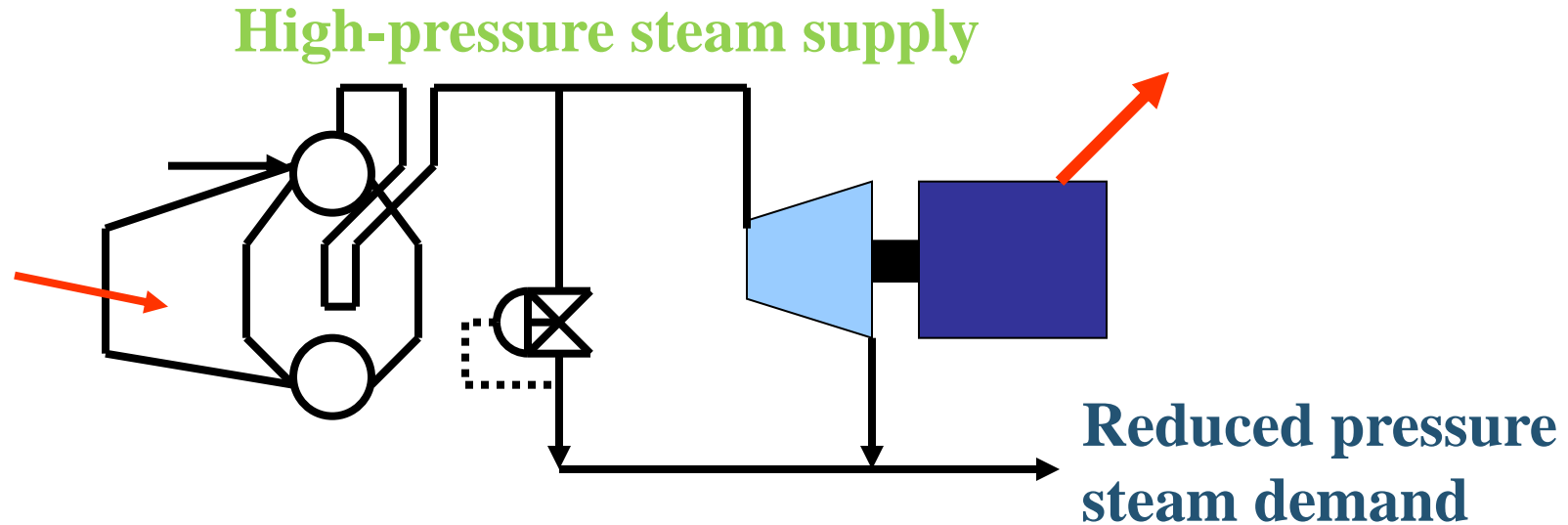
- Turbine balances reduced-pressure header
 - The capacity of the turbine is limitless
 - Any change in low-pressure steam demand will result in a change in steam flow through the turbine

Fixed Turbine – PRV Steam Balance Control



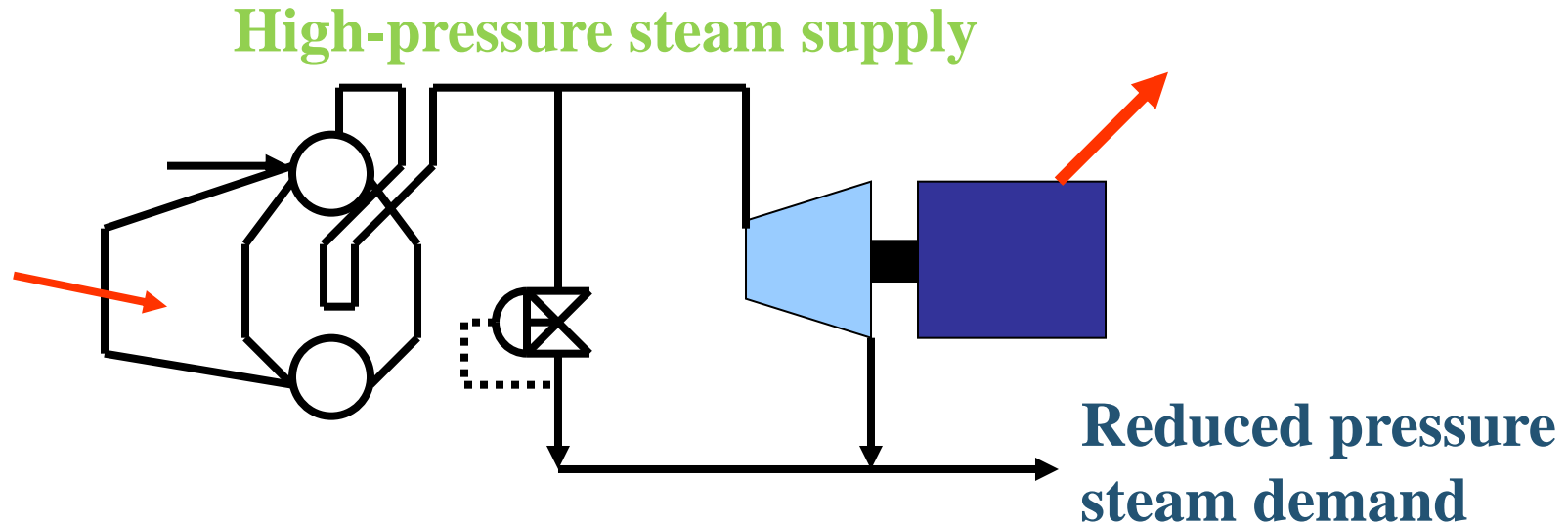
- Turbine is operating based on fixed conditions
 - Steam flow cannot vary through the turbine
 - Any change in low-pressure steam demand will result in a change in steam flow through the PRV
 - Process turbines are typically modeled in this manner

Maximum-Minimum Steam Balance Control



- The turbine can be forced to operate between a minimum and maximum steam flow
 - The PRV will supply additional steam if the turbine capacity limit is reached
 - This provides a realistic limitation based on the capacity of the turbines

Turbine On-Off Control

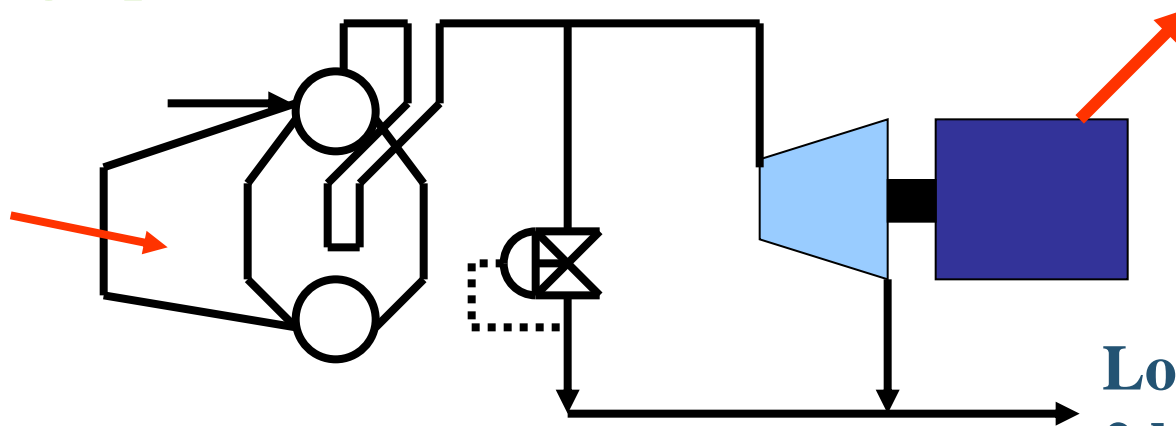


- The turbine can be turned “on” in the input model and “off” in the projects model
 - The turbine can be turned “off” in the input model and “on” in the projects model

Turbine Impact Example

High-pressure steam: 25 bars 375° C

Electrical impact
cost: \$0.14/kWh



Impact boiler: Natural gas (\$0.5/Nm³)
Boiler efficiency: 80%

**Low-pressure steam:
2 bars**

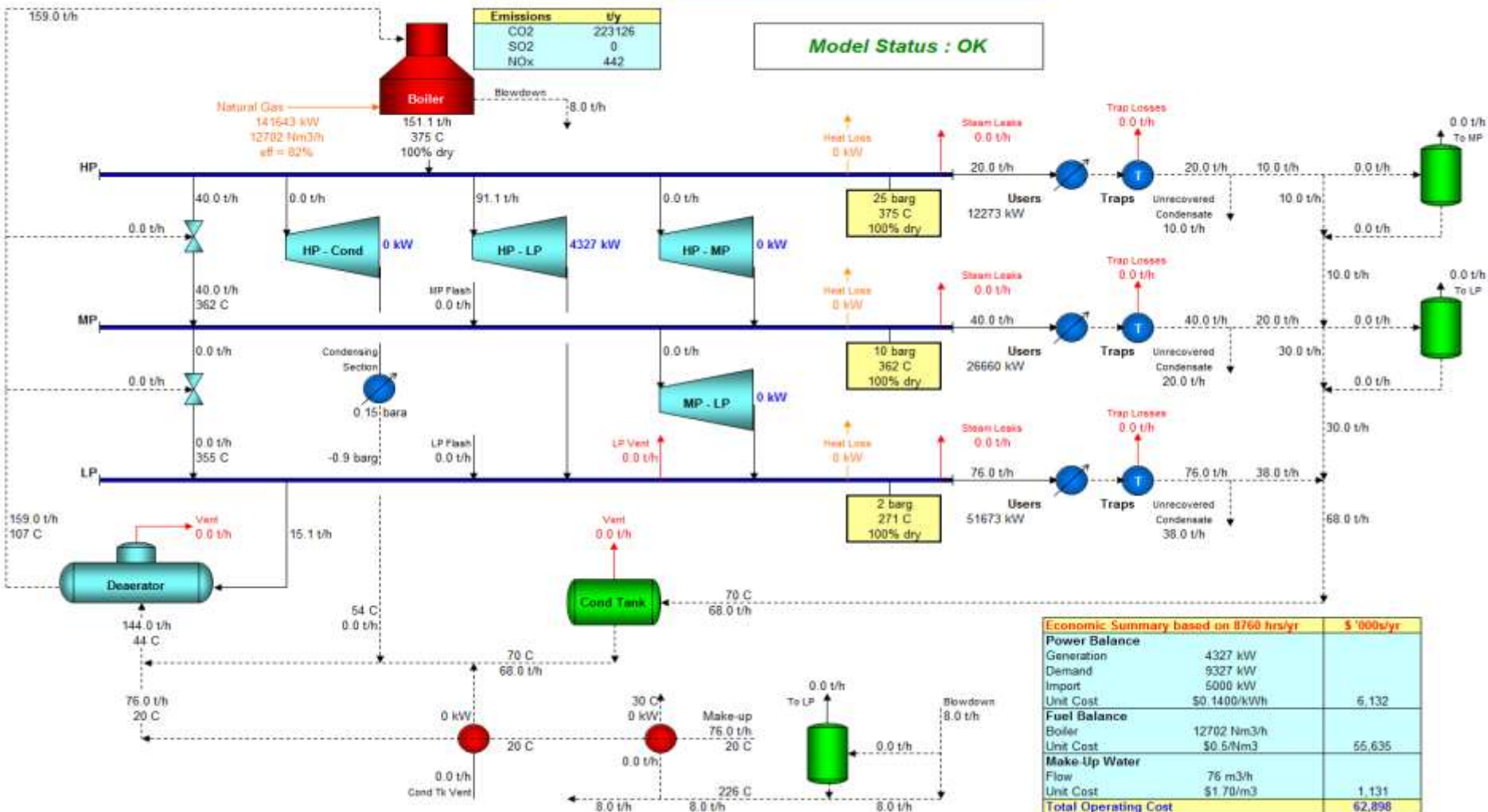
- Open the 3-Header Example System Model and set up the HP-LP turbine with the following configuration
 - Turbine balancing the system
 - Turbine isentropic efficiency = 35%
- Model the economic impact of saving 1 Tph HP and 1 Tph LP steam

Turbine Impact Example Results

Steam System Assessment Tool

SSAT Default 3 Header Metric Model Moldova Ex2

Current Operation



Economic Summary based on 8760 hrs/yr		\$ '000s/yr
Power Balance		
Generation	4327 kW	
Demand	9327 kW	
Import	5000 kW	
Unit Cost	\$0.1400/kWh	6,132
Fuel Balance		
Boiler	12702 Nm3/h	
Unit Cost	\$0.5/Nm3	65,635
Make-Up Water		
Flow	76 m3/h	
Unit Cost	\$1.70/m3	1,131
Total Operating Cost		62,898

Turbine Impact Example Results

Marginal Steam Costs	
(based on current operation)	
HP (\$/t)	46.91
MP (\$/t)	46.91
LP (\$/t)	40.26

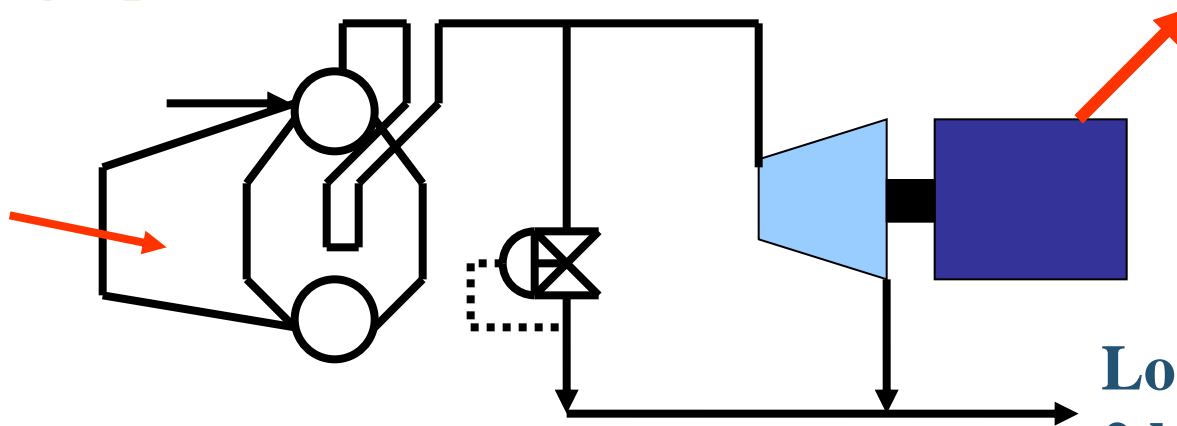
$$CostSavings_{1Tph_HP} = 1.0 \times 8,760 \times 46.91 = \$410,900$$

$$CostSavings_{1Tph_LP} = 1.0 \times 8,760 \times 40.26 = \$356,700$$

Turbine Impact Example

High-pressure steam: 25 bars 375° C

Electrical impact
cost: \$0.14/kWh



Low-pressure steam:
2 bars

Impact boiler: Natural gas (\$0.5/Nm³)
Boiler efficiency: 80%

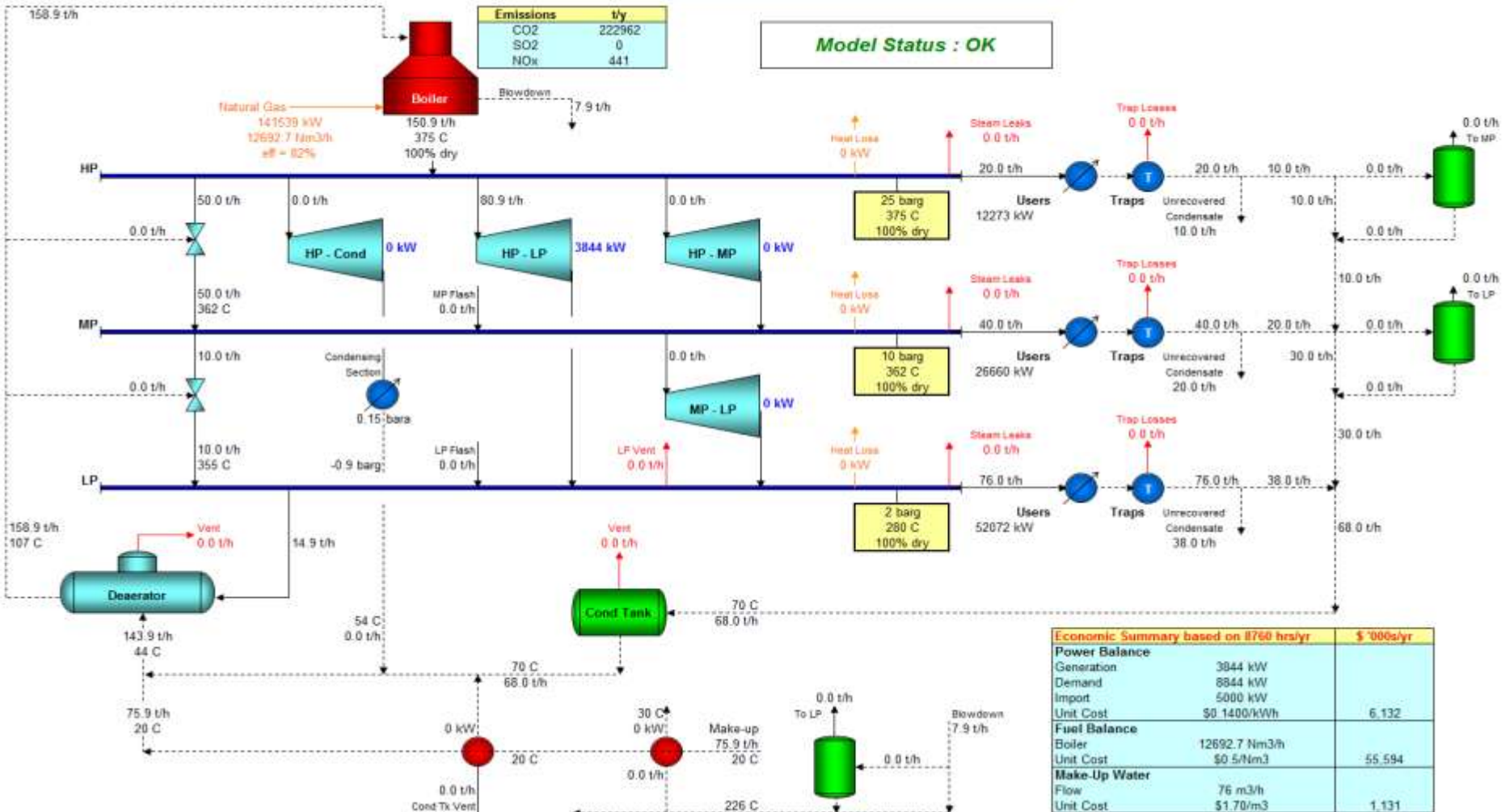
- Open the 3-Header Example System Model and set up the HP-LP turbine with the following configuration
 - MP-LP Pressure Reducing Valve has a flow of ~10 Tph
 - Turbine isentropic efficiency = 35%
- Model the economic impact of saving 1 Tph HP and 1 Tph LP steam

Turbine Impact Example Results

Steam System Assessment Tool

SSAT Default 3 Header Metric Model Moldova Ex2

Current Operation



Turbine Impact Example Results

Marginal Steam Costs	
(based on current operation)	
HP (\$/t)	47.32
MP (\$/t)	47.32
LP (\$/t)	44.63

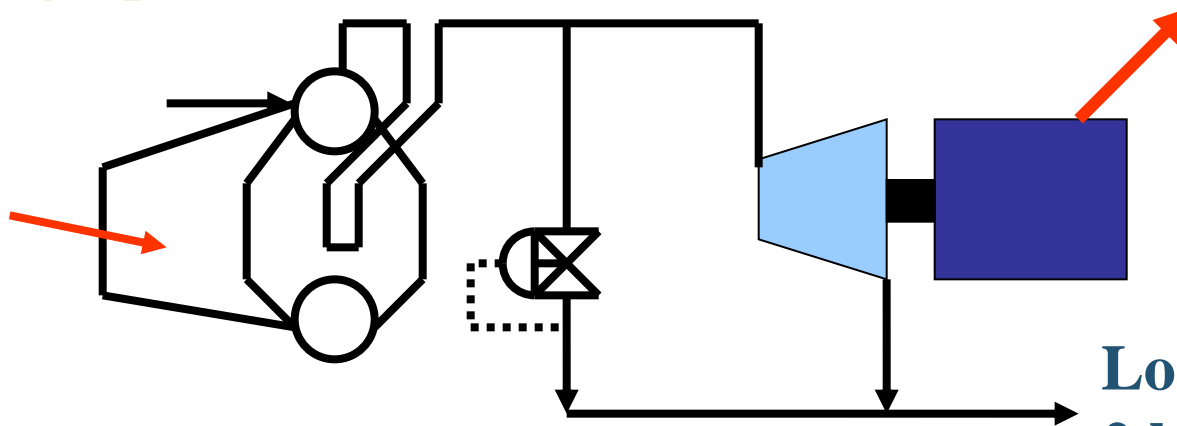
$$CostSavings_{1Tph_HP} = 1.0 \times 8,760 \times 47.32 = \$414,500$$

$$CostSavings_{1Tph_LP} = 1.0 \times 8,760 \times 44.63 = \$391,000$$

Turbine Impact Example

High-pressure steam: 25 bars 375° C

Electrical impact
cost: \$0.14/kWh



Impact boiler: Natural gas (\$0.5/Nm³)
Boiler efficiency: 80%

Low-pressure steam:
2 bars

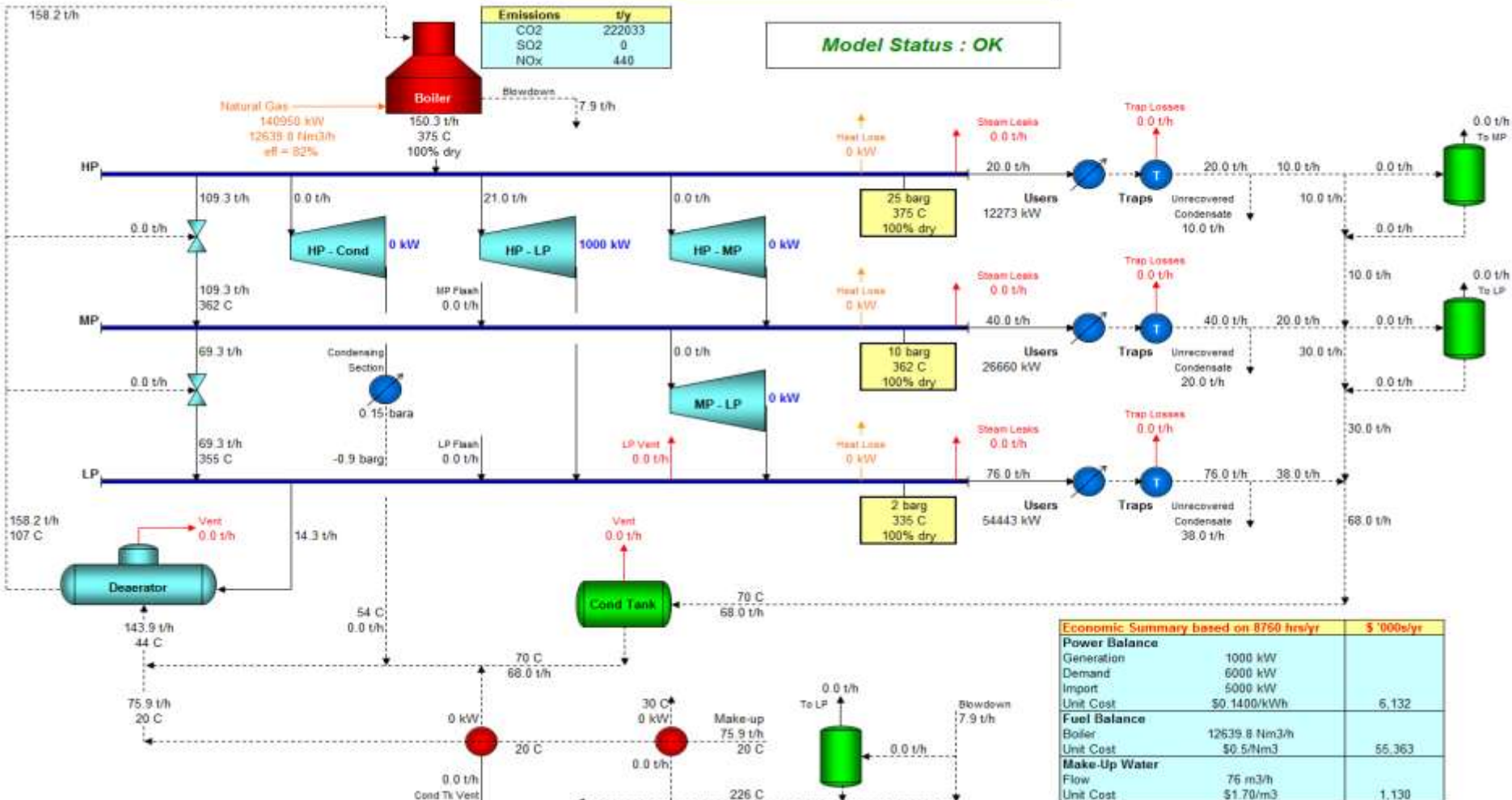
- Open the 3-Header Example System Model and set up the HP-LP turbine with the following configuration
 - Steam turbine flow of ~21.0 Tph
 - Turbine isentropic efficiency = 35%
- Model the economic impact of saving 1 Tph HP and 1 Tph LP steam

Turbine Impact Example Results

Steam System Assessment Tool

SSAT Default 3 Header Metric Model Moldova Ex2

Current Operation



Turbine Impact Example Results

Marginal Steam Costs	
(based on current operation)	
HP (\$/t)	47.34
MP (\$/t)	47.34
LP (\$/t)	46.64

$$CostSavings_{1Tph_HP} = 1.0 \times 8,760 \times 47.34 = \$414,700$$

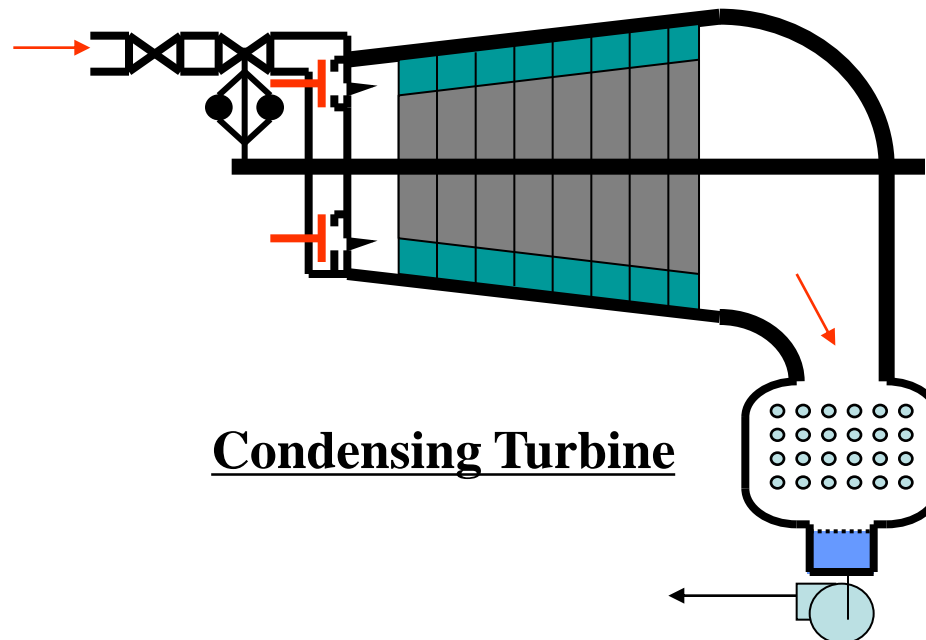
$$CostSavings_{1Tph_LP} = 1.0 \times 8,760 \times 91.77 = \$408,600$$

Backpressure Turbine Economics

- Most industrial systems require thermal energy (not mass flow of steam)
- The turbine will extract energy from the steam and convert it into shaft energy
 - The steam will exit the turbine with a reduced temperature
- The result will be an increased mass flow of steam required to satisfy the thermal demand

Condensing Steam Turbines

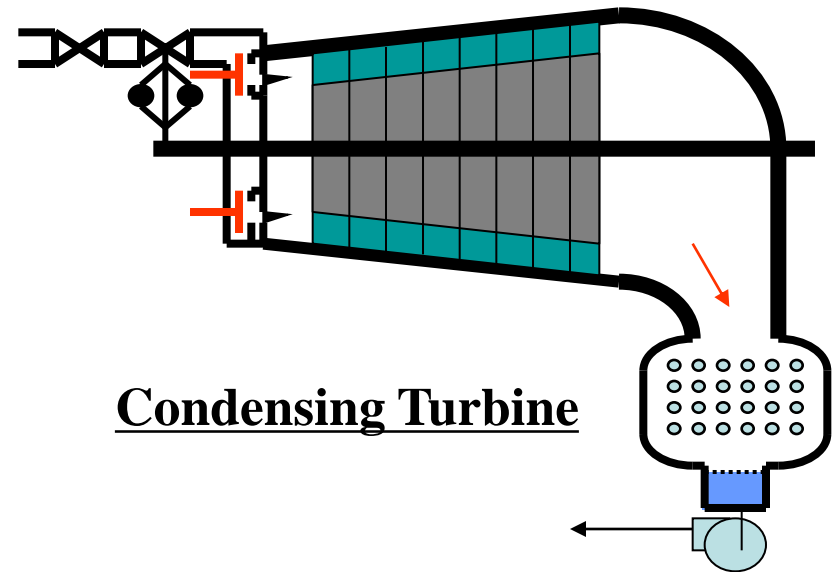
- Condensing steam turbines often operate with a discharge condition of saturated steam
 - Isentropic efficiency is typically determined by
 - Generator output, steam conditions, and steam flow
 - Estimated by manufacturer's data



Condensing Steam Turbines

- Discharge pressure has a significant effect on power production
 - SSAT units of measure are:
 - bara
 - barg
 - Inches of mercury absolute
 - Inches of mercury vacuum

- Condensing turbines are used for
 - Large amount of power generation
 - Driving large mechanical equipment



Condensing Steam Turbines in SSAT

Steam Turbines		
Do you have a steam turbine installed between HP and LP?	Yes	▼
Do you have a steam turbine installed between HP and MP?	No	▼
Do you have a steam turbine installed between MP and LP?	No	▼
Do you have an HP to condensing turbine installed?	Yes	▼

For a **Condensing Turbine**, please define how the turbine operates and then provide supplementary information below:

→ Mode of operation	Option 1 - Define fixed power generation ▼ ←	
→ Option 1 - Fixed power generation	1000 kW	←
Option 2 - Fixed steam flow	25 t/h	

- Condensing steam turbine(s) need to be set up in the “Quick Start” section
 - Use the pull down menu to say “Yes” if a condensing turbine exists
 - Information on control mechanism is also required
- Additional information in the “Site Detail” section

Condensing Steam Turbines in SSAT

Steam Turbines		
Do you have a steam turbine installed between HP and LP?	Yes	▼
Do you have a steam turbine installed between HP and MP?	No	▼
Do you have a steam turbine installed between MP and LP?	No	▼
Do you have an HP to condensing turbine installed?	Yes	▼

For a **Condensing Turbine**, please define how the turbine operates and then provide supplementary information below:

→ Mode of operation	Option 1 - Define fixed power generation ▼ ←	
→ Option 1 - Fixed power generation	1000 kW	←
Option 2 - Fixed steam flow	25 t/h	

- Condensing turbines have two modes of operation
 - Fixed power generation
 - Most process driven equipment operations will have this configuration
 - NOTE: SSAT assumes 100% generator efficiency
 - Fixed steam flow

Condensing Steam Turbines in SSAT

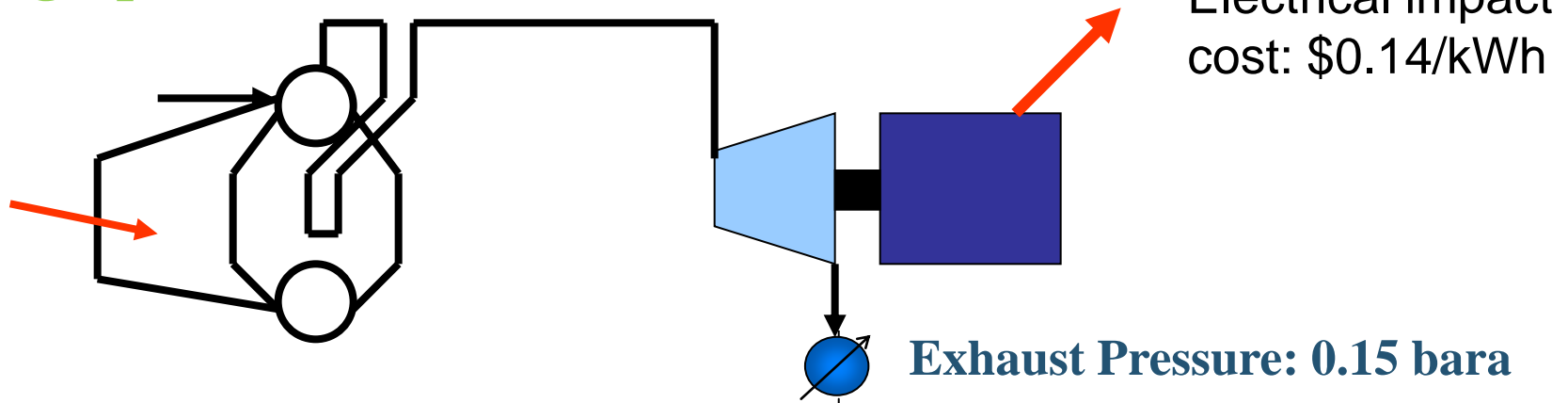
Configure the operation of your HP to Condensing turbine(s) using the options below:

HP to Condensing Steam Turbine(s)	Input Data	Notes/Warnings
→ Isentropic efficiency	65 %	←
Note: If multiple turbines are installed, their data should be combined to allow them to be modeled as a single turbine Note: A generator electrical efficiency of 100% is assumed by the model		
→ Select the units of measure to specify the condenser pressure	bara	▼ ←
→ Condenser pressure (bara)	0.15	←

- Condensing turbine isentropic efficiency is required
 - Manufacturers' data
 - Calculated from steam inlet, flow and power generated
- Condensing turbine outlet (discharge) pressure
 - Can be provided in either of the four units
 - Equivalent to surface condenser pressure

Condensing Turbine Impact Example

High-pressure steam: 25 bars 375° C



Impact boiler: Natural gas: \$1.0/Nm³

Boiler efficiency: 81.7%

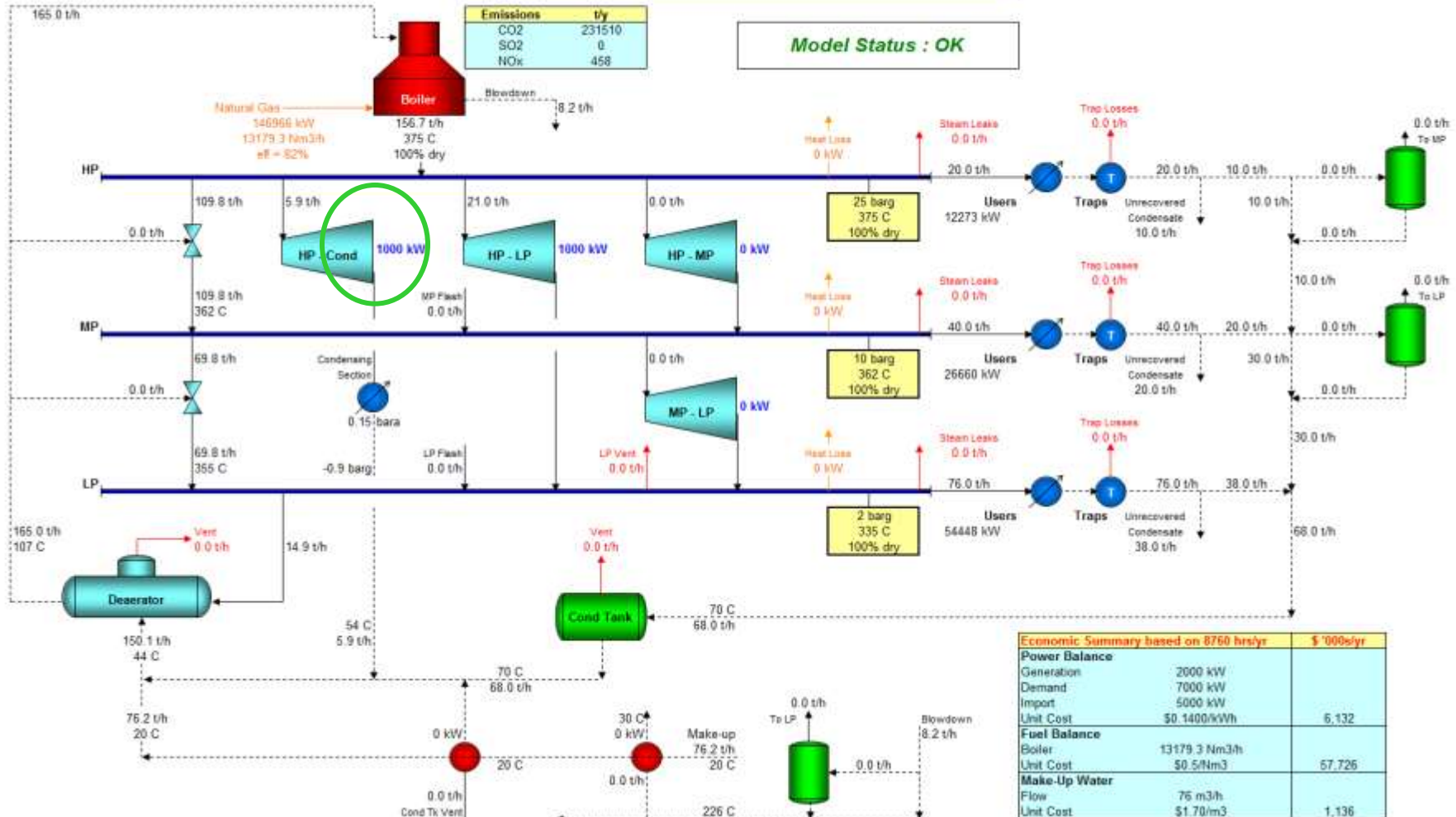
- Open the 3-Header Example System Model and set up the condensing turbine with the following configuration
 - Fixed power generation = 950 kW
 - Generator efficiency = 95%
 - Turbine isentropic efficiency = 65%

Condensing Turbine Example Results

Steam System Assessment Tool

SSAT Default 3 Header Metric Model Moldova Ex2

Current Operation



Example System Model

- All the model “Inputs” are complete
- The 3-header model
 - Closely represents steam flows and steam balance on the headers as would be in the operating case
 - Accurately models the impact (marginal) steam costs of the system
 - DOES NOT represent total utility costs, emissions, etc.
 - NOTE: Impact fuel is used for modeling
 - Is ready to be used to accurately reflect economic impacts of energy saving and optimization opportunities in the steam system
- Make sure it is **SAVED!**